

# TOMAS: Topology Optimization of Multiscale Fluid-Flow Devices using Variational Autoencoders and Super-Shapes

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## Abstract

In this paper, we present a framework for multiscale topology optimization of fluid-flow devices. The objective is to minimize dissipated power, subject to a desired contact-area. The proposed strategy is to design optimal microstructures in individual finite element cells, while simultaneously optimizing the overall fluid flow. In particular, parameterized super-shapes are chosen here to represent microstructures since they exhibit a wide range of permeability and contact area. To avoid repeated homogenization, a finite set of these super-shapes are analyzed *a priori*, and a variational autoencoder (VAE) is trained on their fluid constitutive properties (permeability), contact area, and shape parameters. The resulting differentiable latent space is integrated with a coordinate neural network to carry out a global multi-scale fluid flow optimization. The latent space enables the use of new microstructures that were not present in the original data-set. The proposed method is illustrated using numerous examples in 2D.

**Keywords:** Topology Optimization, Multiscale, Stokes Flow, Variational Auto-encoders, Super-shapes

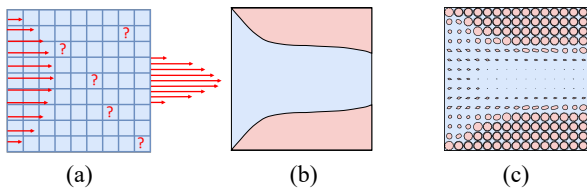
## 1 Introduction

In fluid-flow based topology optimization, the typical objective is to determine the path of least resistance, i.e., least dissipation, within a design domain; see fig. 1(a). When no other constraint is imposed, the path of least resistance is a single connected path [1] as illustrated in fig. 1(b). However, when additional constraints are introduced, the optimal flow-path is typically more complex, and not necessarily a single connected path. One such constraint is the desired fluid-solid contact area,

which plays a crucial role in various applications such as bio-sensors for detecting tumor cells [2], microfluidic devices for cell sorting [3–7], micro-channel heat sinks [8–10], and other microfluidic devices involving heat transfer and mass transportation/mixing mechanisms [11, 12]. In these applications, a minimum fluid-solid contact area is critical for achieving desired performance and functionality. For instance, in bio-sensors for detecting tumor cells, an increased contact area between the fluid and the sensor surface enhances the sensitivity and accuracy of the detection process. Similarly,

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052 in microfluidic devices for cell sorting, the effi-  
 053 ciency of cell capture and separation depends on  
 054 the contact area between the cells and the solid  
 055 surfaces. One approach for enhancing the con-  
 056 tact area is to employ arrays of micro-pillars, as  
 057 suggested in [13, 14], but this can result in a sub-  
 058 stantial increase in dissipated power [15, 16]. A  
 059 more powerful approach is to use multi-scale struc-  
 060 tures, illustrated in fig. 1(c), and the main focus  
 061 of this paper.



070 **Fig. 1:** (a) Fluid-flow problem. (b) Single-scale  
 071 design. (c) Multi-scale design.

## 074 1.1 Single-Scale Fluid Topology 075 Optimization

077 In this section, we briefly review prior research  
 078 on fluid-flow topology optimization (TO). While  
 079 various approaches have been proposed [1], the  
 080 emphasis here is on density-based methods.

081 The field of fluid flow TO was initiated  
 082 by the seminal work of Borrvall and Petersson  
 083 [17]. In their pioneering research, they presented  
 084 an optimal flow layout that minimizes pressure  
 085 drop by employing the Stokes equation along  
 086 with the Brinkman-Darcy equations under low  
 087 Reynolds number conditions. Gersborg-Hansen et  
 088 al. [18] continued this work by presenting applica-  
 089 tions with low Reynolds numbers for microfluidic  
 090 problems and micro-electro-mechanical devices.  
 091 Guest and Prévost [19] introduced the method of  
 092 Darcy–Stokes finite elements to optimize creeping  
 093 fluid flows, producing 0–1 (void-solid) topologies  
 094 without artificial material regions.

095 A novel density-based approach for topology  
 096 optimization of Stokes flow was proposed in [20]  
 097 which addresses convergence issues using fractional-  
 098 order Sobolev spaces for density. Anisotropic mesh  
 099 adaptation is explored in [21] to improve the  
 100 description of solid domains in topology optimiza-  
 101 tion of flow problems. Furthermore, to address  
 102 the bottlenecks of body-fitted mesh evolution

method, reaction–diffusion equation-based fluid  
 topology optimization is explored in [22]. Addition-  
 ally, Wiker et al. [23] explored the use of viscosity  
 as a dependent parameter, providing examples  
 of channels in a tree-shaped structure for pure  
 Darcy problems and mixed Stokes–Darcy flow.  
 The field of fluid flow TO has also been extended  
 to three-phase interpolation models, considering  
 fluid permeability through porous media and  
 impenetrable inner walls using the solid isotropic  
 material with penalization (SIMP) interpolation  
 functions [24]. A Matlab implementation is pre-  
 sented in [25], demonstrating stable low-order  
 discretization of Stokes equations using polygonal  
 finite elements. Parallel computations have been  
 employed for large-scale 2D and 3D Stokes flow  
 problems [26]. In [27], a marker-and-cell method  
 is introduced for large-scale optimization on GPU,  
 utilizing a geometric multigrid preconditioner. A  
 performance-driven optimization of fluidic devices,  
 utilizing parametric boundary descriptions and a  
 differentiable Stokes flow solver is proposed in [28].  
 Furthermore, an anisotropic, differentiable con-  
 stitutive model integrating different phases and  
 boundary conditions within a Stokes model is pre-  
 sented in [29]. Lastly, a phase field approach is  
 introduced in [30] for shape and topology optimiza-  
 tion in Stokes flow, providing a well-posed problem  
 in a diffuse interface setting.

## 1.2 Multiscale Topology Optimization

The methods discussed above result in a single-  
 scale design where all design features are of the  
 same length scale. As discussed earlier, for prob-  
 lems with additional constraints, one must resort  
 to multi-scale TO (MTO), where one designs  
 optimal microstructures in each finite-element  
 cell while simultaneously solving the global flow  
 problem [31]. Several MTO techniques have been  
 proposed for structural and thermal problems.  
 In [32, 33], the authors introduced techniques  
 aimed at finding designs with optimal constitu-  
 tive properties under structural and thermal loads.  
 However, these theoretically optimal designs often  
 lead to extremely small length scales, posing man-  
 ufacturing challenges [34–36]. Additionally, their  
 applicability to fluid problems remains unexplored.  
 A more common approach is to compute optimal  
 microstructures in each cell [31, 37, 38]. While

offering broad design freedom and applicability to various physical phenomena, including fluid flow [31], these methods tend to be computationally expensive [35] since one must carry out homogenization of the evolving microstructures during each step of the optimization process [39].

Homogenization-based optimization can yield disconnected multiscale designs [35]. The process of creating connected multiscale structures from these designs is referred to as de-homogenization [40]. For example, [41, 42] utilized an inverse design methodology to design microchannel fluid flow networks featuring numerous outlets, employing Turing pattern dehomogenization. In [43], dehomogenization based on bioinspired diffusion-generated patterns converted orientation fields into explicit fluid flow channels. Furthermore, in [44], dehomogenization has been used for the inverse design of fiber-reinforced composite with a composite microstructure orientation approach [45]. Other examples of dehomogenization in multiscale topology optimization include the design of shell-infill structures [46], fluid microchannels [47] and inverse design of microreactors [48].

### 1.3 Variations of MTO

To tackle the computational challenges of classic MTO, other techniques have been proposed. For example, graded-MTO (GMTO) [49–56] employs graded variations of pre-selected microstructures. This allows for pre-computation of microstructural properties through offline homogenization before optimization [57, 58]. Machine learning has been utilized to learn the local mechanical properties of multiscale configurations through offline computation [59, 60]. GMTO has been used with both single [61–63] and multiple microstructures [57, 64]. One major limitation of GMTO is that the microstructural shape must be pre-selected prior to optimization. Thus, new microstructure shapes cannot be discovered during optimization. Moreover, these pre-selected shapes are typically graded using a single parameter, which further restricts the variety of microstructures that are generated. To address the aforementioned challenge, a microstructure blending-based multiscale approach has been proposed in [65], which can generate new classes of microstructures. However, the approach requires supplementary parameters

beyond the conventional shape parameters. Moreover, the blending process requires additional steps to impose bounds on the blending operation, to prevent any distorted or invalid shapes in the resulting microstructures. In [64], a set of microstructures were pre-selected, and their size/orientation was optimized. While this slightly increased the design space, it is still limiting and leads to undesirable mixing of microstructures within each cell.

### 1.4 Contributions

In section 2, we propose an alternate and efficient MTO method that uses **variational auto-encoder** (VAE) in combination with super-shapes to compute fluid designs with low dissipation, and desired contact area. In section 3, we demonstrate, using numerical experiments, that this significantly increases in design space. Conclusions and future work are discussed in section 4.

## 2 Proposed Method

### 2.1 Assumptions and Strategy

Consider a design domain  $\Omega^0$  with prescribed flow boundary conditions as illustrated earlier in fig. 1(a). The objective is to compute a multiscale design that minimizes the dissipated power subject to a total contact-area (i.e., perimeter in 2D) constraint. We will assume that it is a low-Reynolds flow, and the fluid is incompressible, i.e., the fluid is governed by Stokes equation:

$$-2\nabla \cdot [\mu \boldsymbol{\epsilon}(\mathbf{u})] + \mathbf{C}_{eff}^{-1} \cdot \mathbf{u} + \nabla p = 0 \text{ in } \Omega^0 \quad (1a)$$

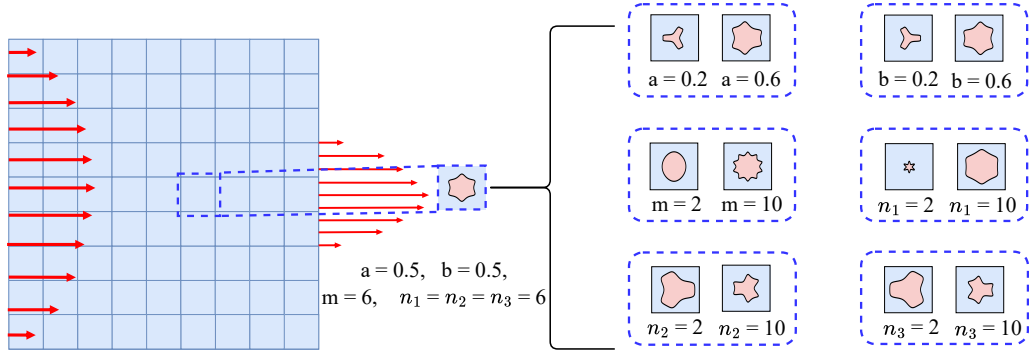
$$\nabla \cdot \mathbf{u} = 0 \text{ in } \Omega^0 \quad (1b)$$

$$\mathbf{u} = \mathcal{G} \text{ over } \partial\Omega^0 \quad (1c)$$

where  $\mathbf{u}$  and  $p$  are the velocity vector and pressure of the fluid,  $\boldsymbol{\epsilon}(\mathbf{u}) = (\nabla \mathbf{u} + \nabla^T \mathbf{u})/2$  represents the rate-of-strain tensor,  $\mathbf{C}_{eff}^{-1}$  denotes the inverse of effective permeability tensor [66, 67] which penalizes the fluid flow in the design domain (for more details see section 2.3). The viscosity  $\mu$  and mass density are assumed to be unity;  $\mathcal{G}$  is the velocity field imposed on  $\partial\Omega^0$ .

The overall strategy is to discretize the domain into finite element cells, and dynamically create optimal microstructures in each cell from a single family of parameterized super-shapes, discussed next.

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**Fig. 2:** A variety of microstructure generated using supershape parameters.

## 2.2 Super-Shapes

Super-shapes, also known as Gielis curves, were introduced by Gielis in 2003 [68] as an extension of super-quadrics. Supershapes have been extended to 3D, the approach involves the spherical product of two super-shapes [69], similar to the method employed for super-quadrics as elaborated in [70]. Unlike super-quadrics that utilize only two parameters, super-shapes incorporate six parameters namely, size:  $a$  and  $b$ , order of rotational symmetry:  $m$ , and curvature:  $n_1$ ,  $n_2$ , and  $n_3$ , where all numbers are assumed to be positive real. Using these six parameters, the super-shape boundary is defined by the set of points:

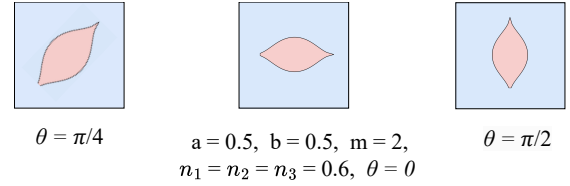
$$(x, y) = (r(\alpha) \cos(\alpha), r(\alpha) \sin(\alpha)) ; 0 \leq \alpha \leq 2\pi \quad (2)$$

where,

$$r(\alpha) = \left[ \left| \frac{1}{a} \cos\left(\frac{m\alpha}{4}\right) \right|^{n_2} + \left| \frac{1}{b} \sin\left(\frac{m\alpha}{4}\right) \right|^{n_3} \right]^{-\frac{1}{n_1}} \quad (3)$$

By varying these six parameters, a variety of shapes can be obtained as illustrated in fig. 2.

In addition to varying these parameters, we allow the shapes be oriented with respect to the x-axis, using an orientation parameter  $\theta$ ; see fig. 3.

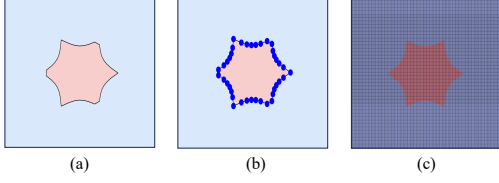


**Fig. 3:** A fish-shaped microstructure oriented at various angles.

The proposed strategy is to find optimal super-shape parameters and orientation, within each cell that minimize the overall dissipated power, subject to a contact-area. A naive approach would entail computing the homogenized constitutive tensors of evolving super-shapes in each cell, during each step of the optimization process. This is once again computationally intractable. Instead, we propose an off-line strategy where a finite set of super-shapes are analyzed, and their characteristics are captured using a VAE [71] (see section 2.4). The resulting decoder (and latent space) is then used for efficient multi-scale optimization. In the remaining sections, the proposed strategy is discussed in detail.

## 2.3 Offline Computation

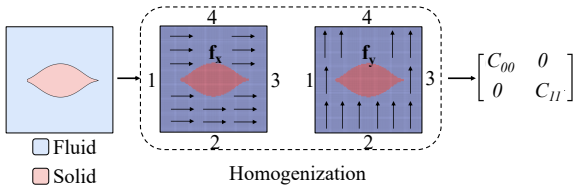
We now describe the process for computing the permeability of any super-shape microstructure. Consider a generic super-shape with a given set of parameters  $\mathbf{M} = \{a, b, m, n_1, n_2, n_3\}$  within a unit cell (i.e.,  $l = 1$ ). The contact-area (i.e., perimeter) and volume fraction are first computed by discretizing the boundary and generating the super-shape density field using the *shapely* library [72], as illustrated in fig. 4(b) and fig. 4(c) respectively.



**Fig. 4:** Generation of super-shape density field using shapely library

To compute the  $2 \times 2$  permeability tensor  $\mathbf{C}$ , the domain is discretized into a mesh of  $150 \times 150$  elements. Then, two Stokes flow problems are solved, subject to unit body forces  $f_x = 1$  and  $f_y = 1$ , as illustrated in fig. 5. The boundary conditions involve coupling boundaries 1 and 3 through periodic conditions for velocity and pressure, as well as coupling boundaries 2 and 4 similarly; see [73]. The velocities obtained from solving the problem with  $f_x = 1$  are denoted as  $u_0(x, y)$  and  $v_0(x, y)$ , while those obtained from solving the problem with  $f_y = 1$  are denoted as  $u_1(x, y)$  and  $v_1(x, y)$ . Since  $u_1(x, y)$  and  $v_0(x, y)$  are nearly orthogonal to the bulk flow directions, the off-diagonal terms of the permeability tensor are three orders of magnitude smaller than the diagonal terms and can be neglected [74]. Thus, the permeability tensor  $\mathbf{C}$  is computed as follows [73, 75, 76]:

$$\mathbf{C} = \begin{bmatrix} C_{00} & 0 \\ 0 & C_{11} \end{bmatrix} = \frac{1}{|V|} \begin{bmatrix} \int_V u_0 dV & 0 \\ 0 & \int_V v_1 dV \end{bmatrix} \quad (4)$$



**Fig. 5:** Offline homogenization.

where the volume  $V$  of the unit cell is unity. Note that the orientation is taken into account

through the following tensor operation to determine the transformed permeability tensor [75]:

$$\mathbf{C}_{trans} = \begin{bmatrix} \cos \theta & -\sin \theta \\ \sin \theta & \cos \theta \end{bmatrix} \begin{bmatrix} C_{00} & 0 \\ 0 & C_{11} \end{bmatrix} \begin{bmatrix} \cos \theta & -\sin \theta \\ \sin \theta & \cos \theta \end{bmatrix}^T \quad (5)$$

As the effective permeability  $\mathbf{C}_{eff}$  in equation eq. (1) is scale-dependent, one can scale  $\mathbf{C}_{trans}$  according to the unit-cell size  $l$  of the problem, expressed as  $\mathbf{C}_{eff} = l^2 \mathbf{C}_{trans}$ .

For numerical homogenization, a Brinkman penalization of zero is applied for the fluid phase and  $10^6$  for the solid phase. For further details on the numerical homogenization methodology employed in this study, please see [73]. A random set of 7000 samples of super-shapes are analyzed using the above process, where parameter instances are generated using a uniform random distribution [77] as follows:  $0.05 \leq a, b \leq 0.75$ ,  $1 \leq m \leq 22$  and  $0.5 \leq n_1, n_2, n_3 \leq 10$ . The results from the offline computation are then analyzed using variational autoencoders, discussed next.

## 2.4 Variational Auto-Encoders

Variational auto-encoders (VAEs) are a type of generative model that leverages probabilistic encoding and decoding techniques to compress input data into a lower-dimensional latent space [71, 78, 79]. One of the key advantages of VAEs, as opposed to other encoding methods, is their ability to generate new samples that resemble the original input data [79]. For instance, VAEs have been successfully employed to generate novel microstructures from image databases [80]. Another important feature of VAEs is the creation of continuous and differentiable latent space. This allows for gradient-based optimization, enabling efficient exploration of the latent space. This is particularly valuable in applications such as reliability-based TO [81]. Finally, unlike linear dimensionality reduction techniques such as principal component analysis (PCA), VAEs can learn complex non-linear relationships between the input and the reduced dimensional space [82].

The proposed VAE architecture, depicted in fig. 6, consists of several essential components:

1. Firstly, the input  $\Psi$  is ten-dimensional representing six shape parameters  $\mathbf{M}$ , two permeability components ( $C_{00}$  and  $C_{11}$ ), contact area  $\Gamma$  and volume fraction  $v_f$ .

- 256 2. The encoder  $E$ , following along [83], is a fully-  
 257 connected network consisting of two hidden  
 258 layers, each containing 600 neurons with a  
 259 LeakyReLU activation functions [84].  
 260 3. The proposed VAE uses a two-dimensional  
 261 latent space ( $z_1$  and  $z_2$ ).  
 262 4. Additionally, a decoder  $D$  is constructed  
 263 with two hidden layers, each containing 600  
 264 neurons.  
 265 5. The output consists of the same ten properties:  
 266 shape parameters  $\hat{\mathbf{M}}$ , permeability compo-  
 267 nents  $\hat{\mathbf{C}} \equiv (\hat{C}_{00}$  and  $\hat{C}_{11})$ , contact area  $\hat{\Gamma}$  and  
 268 volume fraction  $\hat{v}_f$ . They can be combined as  
 269  $\hat{\Psi} \equiv (\hat{a}, \hat{b}, \hat{m}, \hat{n}_1, \hat{n}_2, \hat{n}_3, \hat{C}_{00}, \hat{C}_{11}, \hat{\Gamma}, \hat{v}_f)$ .

270 Note that the reconstruction will not be exact.  
 271 One objective is to minimize the difference between  
 272 the output and input [85]. This involves opti-  
 273 mizing the weights associated with the encoder  
 274 and decoder. Additionally, the latent space is  
 275 constrained to approximate a Gaussian distribu-  
 276 tion  $\mathcal{N}(0, I)$  through a KL divergence loss term  
 277 expressed as  $\text{KL}(z|\mathcal{N})$  [71]. This ensures that sim-  
 278 ilar shapes are clustered together in the latent  
 279 space. Thus, the overall VAE loss function can be  
 280 formulated as:

$$281 \quad L_v = \|\Psi - \hat{\Psi}\|_2 + \beta \text{KL}(z|\mathcal{N}) \quad (6)$$

282 Here,  $\beta$  is set to  $10^{-7}$  [85]. To achieve a stable  
 283 convergence, the geometric parameters, contact  
 284 area, and volume fraction are normalized linearly  
 285 between 0 and 1, while the permeability compo-  
 286 nents are scaled logarithmically due to significant  
 287 variation in magnitude.

## 291 2.5 Latent Space

292 Once the latent space has been constructed, the  
 293 trained decoder  $D^*$  can be used to generate  
 294 super-shape parameters and properties via  $\hat{\Psi} =$   
 295  $D^*(z_1, z_2)$  for all points within the latent space.  
 296 The generated latent space has the following  
 297 features:

- 298 1. **Generation of new microstructures:**  
 299 Although the data set used to train the  
 300 decoder is discrete, the resulting latent space  
 301 is continuous. This continuous representa-  
 302 tion facilitates a meaningful exploration of  
 303 microstructure configurations throughout the  
 304 latent space. For example in fig. 7, while A,  
 305 B, E, and F depict points present in the data  
 306

set, points C, D, G, and H are generated  
 by the trained decoder, with correspond-  
 ing microstructures. The percentage error  
 between the actual microstructure properties  
 obtained from numerical homogenization and  
 the reconstructed data for points both within  
 (A, B, E, and F) and outside (C, D, G, and  
 H) the training dataset are summarized in  
 table 1. We found that the percentage errors  
 for both sets of points are comparable.

2. **Differentiable Latent Space:** The latent  
 space is differentiable in that derivatives  
 such as  $\frac{\partial \hat{\Psi}}{\partial z_1}$ , can be computed analyti-  
 cally using back-propagation. This enables  
 gradient-based optimization.

Points	$\Delta C_{00}\%$	$\Delta C_{11}\%$	$\Delta v_f\%$	$\Delta \Gamma\%$
A	0.42	3.70	0.232	1.20
B	1.84	3.63	0.291	3.30
C	3.50	2.80	0.20	0.76
D	1.80	3.60	1.60	2.60
E	0.43	3.70	0.60	2.03
F	0.122	4.05	0.947	1.24
G	1.83	3.63	1.64	2.65
H	2.05	0.322	2.38	2.63

Table 1: Prediction accuracy for points in  
 fig. 7

## 2.6 Global Fluid Flow Analysis

We are now ready to address global fluid flow  
 analysis. Here, a quadrilateral Q2-Q1 (quadratic  
 velocity/linear pressure) element belonging to the  
 class of the Taylor-Hood elements is used. The  
 elemental stiffness matrix  $\mathbf{K}_e$  and degrees of free-  
 dom vector  $\mathbf{S}_e$  for the governing equation (see  
 section 2.1) are given by (see [25] for details):

$$\mathbf{K}_e = \begin{bmatrix} \mathbf{A}_e & \mathbf{B}_e & \mathbf{0} \\ \mathbf{B}_e^T & \mathbf{0} & \mathbf{h}_e \\ \mathbf{0} & \mathbf{h}_e^T & \mathbf{0} \end{bmatrix}, \quad \mathbf{S}_e = \begin{bmatrix} \mathbf{U}_e \\ \mathbf{P}_e \\ \lambda \end{bmatrix} \quad (7)$$

where

$$\mathbf{A}_e = \mathbf{A}_e^\mu + \mathbf{C}_{eff,e}^{-1} \mathbf{A}_e^\alpha \quad (8a)$$

$$[\mathbf{A}_e^\mu]_{ij} = \int_{\Omega_e} 2\mu \epsilon(\mathbf{N}_i) : \epsilon(\mathbf{N}_j) d\Omega \quad (8b)$$

$$[\mathbf{A}_e^\alpha]_{ij} = \int_{\Omega_e} \mathbf{N}_i \mathbf{N}_j d\Omega \quad (8c)$$

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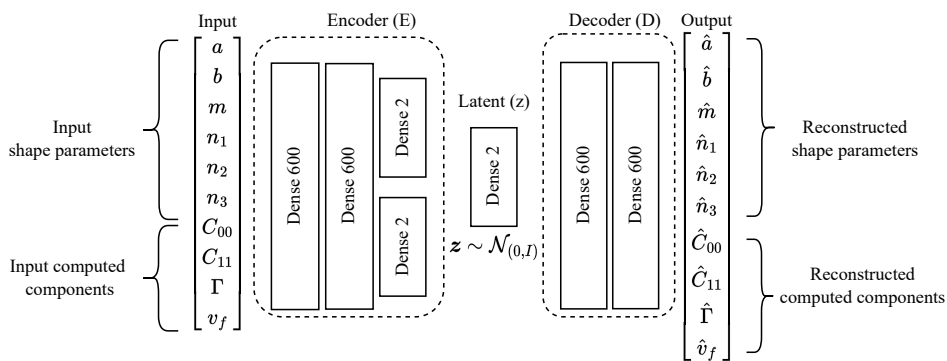


Fig. 6: Proposed VAE network.

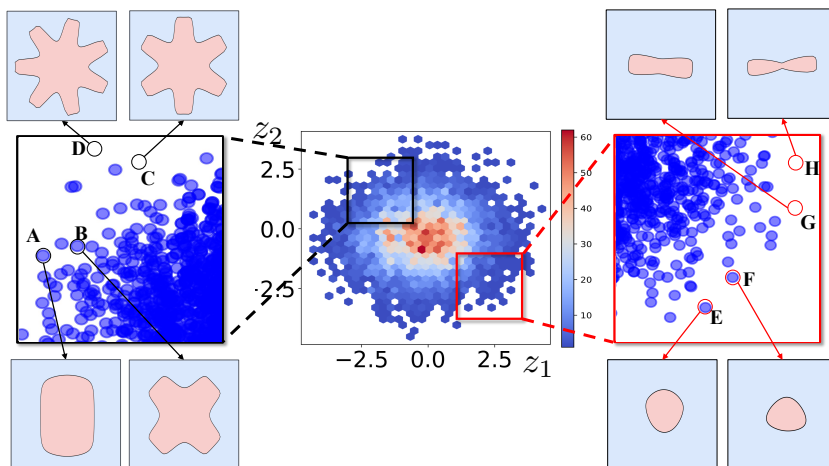


Fig. 7: The latent space density distribution and scattered plots in the insets reveal both microstructures existing in the dataset and new microstructures generated by the VAE that are not originally present in the dataset.

$$[\mathbf{B}_e]_{ij} = \int_{\Omega_e} \mathbf{L}_j \nabla \cdot \mathbf{N}_i d\Omega \quad (8d)$$

$$[\mathbf{h}_e]_i = \int_{\Omega_e} \mathbf{L}_i d\Omega \quad (8e)$$

Here,  $\mathbf{N}_i$  and  $\mathbf{L}_i$  are the velocity and pressure basis functions,  $\mathbf{U}_e$  and  $\mathbf{P}_e$  represent elemental velocity and pressure degrees of freedom respectively and  $\mathbf{C}_{eff,e}$  is the design dependent effective element permeability matrix (section 2.3).

is defined in section 2.1. To ensure a unique definition of the pressure field, a zero mean condition ( $\mathbf{h}_e^T \mathbf{P} = 0$ ) is imposed by incorporating a Lagrange multiplier, denoted as  $\lambda$  (for details see [25, 86]). The individual elemental  $\mathbf{K}_e$  and  $\mathbf{S}_e$  matrices are assembled to construct the global stiffness matrix  $\mathbf{K}$  and degrees of vector  $\mathbf{S}$  respectively. We then solve the equation  $\mathbf{K}\mathbf{S} = \mathbf{f}$ , wherein the vector  $\mathbf{f}$  represents the boundary conditions applied. This solution determines the unknown degrees of freedom in  $\mathbf{S}$ .

## 2.7 Design Variables, Objective and Constraints

Finally, the optimization framework comprises of the following:

**Design Variables:** The design variables associated with each element are denoted by  $\zeta_e = \{z_{1,e}, z_{2,e}, \theta_e\}$ , where  $z_{1,e}$  and  $z_{2,e}$  are the two latent space variables, and  $\theta_e$  is the orientation of the super-shape. The values  $z_1$  and  $z_2$  are constrained to lie within  $[-3, 3]$ , and the orientation parameter is constrained as  $0 \leq \theta \leq 2\pi$ . The entire set of design variables is denoted by  $\bar{\zeta} = \{\zeta_1, \zeta_2, \dots, \zeta_{N_e}\}$ .

**Objective:** The objective is to minimize the dissipated power given by [17, 25]:

$$J(\bar{\zeta}) = \sum_{e=1}^{N_e} \frac{1}{2} U_e^T [A_e^\mu + A_e^\alpha C_{eff,e}^{-1}] U_e \quad (9)$$

**Contact-Area Constraint:** The contact area  $\hat{\Gamma}_e$  of each microstructure is reconstructed using the decoder, and the following global constraint is imposed:

$$g_\Gamma(\bar{\zeta}) \equiv 1 - \frac{\sum_{e=1}^{N_e} \hat{\Gamma}_e}{\Gamma_{min}} \leq 0 \quad (10)$$

where  $\Gamma_{min}$  is the lower bound on the contact area and  $N_e$  represents the number of elements in the design domain.

**Volume Constraint:** Instead of imposing a contact area constraint, one can impose a volume constraint:

$$g_V(\bar{\zeta}) \equiv \frac{\sum_{e=1}^{N_e} \hat{v}_{f,e}}{N_e v_{max}} - 1 \leq 0 \quad (11)$$

where,  $\hat{v}_{f,e}$  is the fluid volume fraction, and  $v_{max}$  is the upper bound on the volume fraction.

## 2.8 Multiscale Optimization Problem

Consequently, one can pose the multiscale problem in a finite-element setting as:

$$\min_{\bar{\zeta}=\{\zeta_1, \zeta_2, \dots, \zeta_{N_e}\}} J(\bar{\zeta}) \quad (12a)$$

$$\text{subject to} \quad K(\bar{\zeta})S = f \quad (12b)$$

$$g_\Gamma(\bar{\zeta}) \leq 0 \quad (12c)$$

$$\text{(or)} \quad g_V(\bar{\zeta}) \leq 0 \quad (12d)$$

$$\text{with} \quad -3 \leq z_{e,0}, z_{e,1} \leq 3, \forall e \quad (12e)$$

$$0 \leq \theta_e \leq 2\pi, \forall e \quad (12f)$$

To solve the above optimization problem, optimization techniques such as the method of moving asymptotes [87] or optimality criteria [88] can be used. However, we use neural networks (NN) for optimization [89] due to several advantages. [Most NN implementations support automatic differentiation \[90\], which enables seamless gradient calculations. They can also capture highly non-linear behavior with very few design variables.](#)

## 2.9 Optimization using a Neural Network

The proposed neural-network (NN) architecture for global optimization is illustrated in fig. 8, and it consists of the following entities:

- Input Layer:** The input to the NN are points  $\mathbf{x} \in \mathbf{R}^2$  within the domain  $\Omega^0$ . Although these points can be arbitrary, they correspond here to the center of the elements.
- Fourier Projection:** The sampled points from the Euclidean domain are directed through a frequency space, associated with a frequency range  $\mathbf{F}$ . Prior research [91, 92] indicates that implicit coordinate-based neural networks are biased to lower frequency components of the target signal. To address this issue and speed up convergence, a Fourier projection layer is integrated before the standard activation layers [93].
- Hidden Layers:** The hidden layers consist of a series of fully connected LeakyReLU activated neurons, LeakyReLU is a differentiable function, as opposed to ReLU, and is therefore preferred in this work [94]. In particular, the neural network used here consists of two hidden layers, each activated with the LeakyReLU function and each layer has 20 neurons.
- Output Layer:** The output layer consists of 3 neurons corresponding the design variables for each element  $\zeta_e = \{z_1(\mathbf{x}), z_2(\mathbf{x}), \theta(\mathbf{x})\}$ . The output neurons are activated by a Sigmoid



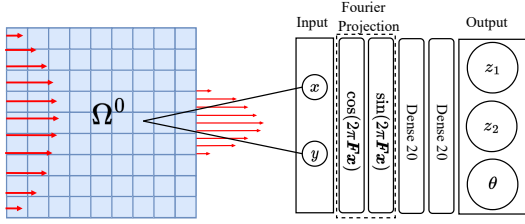


Fig. 8: Topology optimization network.

function  $\sigma(\cdot)$ . The neurons associated with the latent space variables are scaled as  $z_i \leftarrow -3 + 6\sigma(z_i)$  to retrieve values in the range of a standard Gaussian Normal distribution. Further, the output neuron associated with the orientation is scaled as  $\theta \leftarrow 2\pi\sigma(\theta)$ . Thus, the box constraints in eq. (12e) and eq. (12f) are not needed.

5. **NN Design Variables:** The weights and bias associated with the NN, denoted by the  $\mathbf{w}$ , now become the primary design variables, i.e., we have  $z_1(\mathbf{x}; \mathbf{w})$ ,  $z_2(\mathbf{x}; \mathbf{w})$  and  $\theta(\mathbf{x}; \mathbf{w})$ . The weights in the network are initialized using Xavier weight initialization [95] with a seed value of 77.
6. **Optimizer:** The Adam optimizer is used with a learning rate of  $4 \cdot 10^{-3}$ . The optimization process is set to run for a maximum of 300 iterations (epochs). To ensure convergence, the optimization monitors the change in loss ( $\Delta L_c^*$ ) (see eq. (14)) with a threshold of  $10^{-5}$ .

Thus, eq. (12) reduces to:

$$\underset{\mathbf{w}}{\text{minimize}} \quad J(\mathbf{w}) \quad (13a)$$

$$\text{subject to} \quad \mathbf{K}(\mathbf{w})\mathbf{S} = \mathbf{f} \quad (13b)$$

$$g_\Gamma(\mathbf{w}) \equiv 1 - \frac{\sum_{e=1}^{N_e} \hat{\Gamma}_e(\mathbf{w})}{\Gamma_{min}} \leq 0 \quad (13c)$$

$$\text{(or)} \quad g_V(\mathbf{w}) \equiv \frac{\sum_{e=1}^{N_e} (\hat{v}_{f,e}(\mathbf{w}))}{N_e v_{max}} - 1 \leq 0 \quad (13d)$$

Since neural networks are designed to minimize an unconstrained loss function, we convert the constrained minimization problem into a loss function minimization by employing the penalty scheme

[96]. Specifically, the loss function is defined as:

$$L_T(\mathbf{w}) = \frac{J(\mathbf{w})}{J^0} + \gamma g(\mathbf{w})^2 \quad (14)$$

where the parameter  $\gamma$  is updated during each iteration, making the enforcement of the constraint ( $g$ ) stricter as the optimization progresses. The constraint penalty in the current framework starts with an initial value of  $\gamma = 1$ . and is incremented by  $\Delta\gamma = 0.1$  after every epoch.  $J^0$  represents the initial iteration's objective value, which serves as a scaling factor for the objective function.

Thus, the overall framework is illustrated in fig. 9.

## 2.10 Sensitivity Analysis

A critical ingredient in gradient-based optimization is the sensitivity, i.e., derivative, of the objective and constraint(s) with respect to the optimization parameters. Typically the sensitivity analysis is carried out manually. For example, the derivatives of the objective function are typically expressed as follows where each term is computed manually:

$$\frac{\partial L_T}{\partial \mathbf{w}} = \left[ \frac{\partial L_T}{\partial J} \frac{\partial J}{\partial \mathbf{u}} \frac{\partial \mathbf{u}}{\partial \mathbf{K}} \left( \frac{\partial \mathbf{K}}{\partial \hat{\mathbf{C}}} \frac{\partial \hat{\mathbf{C}}}{\partial \mathbf{w}_D^*} \frac{\partial \mathbf{w}_D^*}{\partial \mathbf{z}} \frac{\partial \mathbf{z}}{\partial \mathbf{w}} + \frac{\partial \mathbf{K}}{\partial \theta} \frac{\partial \theta}{\partial \mathbf{w}} \right) + \frac{\partial L_T}{\partial g_\Gamma} \frac{\partial g_\Gamma}{\partial \hat{\Gamma}} \frac{\partial \hat{\Gamma}}{\partial \mathbf{w}_D^*} \frac{\partial \mathbf{w}_D^*}{\partial \mathbf{z}} \frac{\partial \mathbf{z}}{\partial \mathbf{w}} \right] \quad (15)$$

This can be laborious and error-prone, especially for non-trivial objectives. Here, by expressing all our computations including computing the permeability tensors, stiffness matrix, FEA, objectives, and constraints in PyTorch [90], we use the NN's automatic differentiation (AD) capabilities to completely automate this step [97]. In other words, only the forward expressions need to be defined, and all required derivatives are computed to machine precision by PyTorch computing library. PyTorch automatically handles computing the gradients by constructing a computational graph, with the weights of the NN serving as the leaf tensors [98]. This graph encompasses all the operations depicted in the flow chart in fig. 9. By invoking the gradient function with respect to the loss, PyTorch effectively calculates the sensitivities eq. (15), including the adjoint computation  $\frac{\partial \mathbf{u}}{\partial \mathbf{K}}$ .

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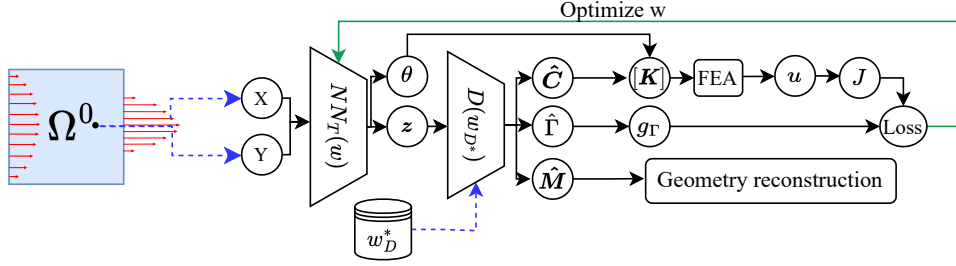


Fig. 9: Optimization flowchart.

## 2.11 Algorithms

The three algorithms used here are summarized in algorithm 1, algorithm 2, and algorithm 3. In the first algorithm, the primary objective is to generate a set of microstructures and their properties  $\Psi$ .

---

**Algorithm 1** Generate Microstructure properties data-set

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- 1: **procedure** DATAGEN
  - 2:  $M \rightarrow \Gamma, v_f, C_{00}, C_{11}$  ▷ Offline  
Computation section 2.3
  - 3:  $M, \Gamma, v_f, C_{00}, C_{11} \rightarrow \Psi$  ▷ Data-set  
creation
  - 4: **end procedure** ▷ Output: Microstructure  
data-set
- 

In algorithm 2, using  $\Psi$ , the VAE is trained to produce a lower-dimensional latent space.

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**Algorithm 2** Encode Microstructure properties

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- 1: **procedure** MSTRENCODE( $\Psi, E, D$ ) ▷ Input:  
Training data, encoder and decoder
  - 2: epoch = 0 ▷ iteration counter
  - 3: **repeat** ▷ VAE training
  - 4:  $E(\Psi) \rightarrow z$  ▷ Forward prop.encoder
  - 5:  $D(z) \rightarrow \hat{\Psi}$  ▷ Forward prop.decoder
  - 6:  $\{\Psi, \hat{\Psi}, z\} \rightarrow L_V$  ▷ VAE loss eq. (6)
  - 7:  $w, \nabla L_V \rightarrow w$  ▷ Adam optimizer step;  
update weights
  - 8: epoch ++
  - 9: **until**  $\|\Delta L_V\| < \Delta \hat{L}_V$  or epoch < max\_epoch ▷ check for convergence
  - 10: **end procedure** ▷ Output: Trained Decoder
- 

Once the training is complete, the encoder  $E$  is discarded, and the decoder  $D$  is retained. The

main optimization algorithm is summarized in algorithm 3. First, the domain  $\Omega^0$  is discretized for finite element analysis, and the stiffness matrix components are computed (line 3). The mesh is sampled at the center of each element (line 4); these serve as inputs to the NN. The penalty parameter  $\gamma$  and NN weights  $w$  are initialized [95] (line 5).

In the main iteration, the design variables  $\bar{\zeta}$  are computed using the NN (line 7). Then the latent space variables serve as input to the trained decoder  $D^*$ , followed by the computation of the microstructural geometric properties  $\hat{M}$ , permeability components  $\hat{C}$  and contact area  $\hat{\Gamma}$  (or, alternately, the volume fraction) for each element (line 8). The reconstructed permeability components from the decoder along with the orientation from the NN are used to calculate the effective permeability tensor (line 9). The effective permeability is used to construct the stiffness matrix and to solve for the velocity and pressure (lines 10-11). Then the objective and contact area (or volume) constraint are computed (lines 12 - 13), leading to the loss function (line 14). The sensitivities are computed in an automated fashion (line 15). The weights  $w$  are then updated using the Adam optimization scheme (line 16). Finally, the penalty parameters are updated (line 17). The process is repeated until termination, i.e., until the relative change in loss is below a certain threshold or the iterations exceed a maximum value.

## 3 Numerical Experiments

In this section, we conduct several experiments to demonstrate the proposed framework. All experiments were conducted on a MacBook M2 Air, using the PyTorch library [90] in Python. [Although some hyperparameter tuning was necessary initially \(specifically, learning rate, neural network](#)

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**Algorithm 3** Fluid Topology Optimization
 

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```

1: procedure TOPOPT( $\Omega^0$ , BC,  $\Gamma_{min}$ ,  $D^*$ )  $\triangleright$ 
   Input: Design domain, boundary conditions,
   area constraint, and trained decoder
2:    $\Omega^0 \rightarrow \Omega_h^0$  section 2.6
3:    $\Omega_h^0 \rightarrow \mathbf{A}^\mu, \mathbf{A}^\alpha, \mathbf{B}, \mathbf{h}$  eq. (8)
4:    $\mathbf{x} = \{x_e, y_e\}_{e \in \Omega_h^0}$   $\mathbf{x} \in \mathbb{R}^{n_e \times 2}$ 
5:   epoch = 0;  $\gamma = \gamma_0$ ;  $\mathbf{w} = \mathbf{w}_0$   $\triangleright$  initialization
6:   repeat  $\triangleright$  optimization (Training)
7:      $NN(\mathbf{x}; \mathbf{w}) \rightarrow \bar{\mathbf{z}}(\mathbf{x}), \bar{\boldsymbol{\theta}}(\mathbf{x})$   $\triangleright$  section 2.9
8:      $D^*(\bar{\mathbf{z}}(\mathbf{x}; \mathbf{w})) \rightarrow \hat{\mathbf{M}}(\mathbf{x}), \hat{\mathbf{C}}$   $\triangleright$  fwd prop
9:      $\hat{\mathbf{C}}_{00}(\mathbf{x}), \hat{\mathbf{C}}_{11}(\mathbf{x}), \theta(\mathbf{x}) \rightarrow \mathbf{C}_{eff}(\mathbf{x})$   $\triangleright$ 
   eq. (5)
10:     $\mathbf{C}_{eff}(\mathbf{x}) \rightarrow \mathbf{K}, \mathbf{f}$  eq. (7)
11:     $\mathbf{K}, \mathbf{f} \rightarrow \mathbf{S}$   $\triangleright$  solve eq. (13b)
12:     $\mathbf{K}, \mathbf{S} \rightarrow J$   $\triangleright$  Objective, eq. (13a)
13:     $\hat{\Gamma}, \Gamma_{min} \rightarrow g_\Gamma$  eq. (10)
14:     $J, g_\Gamma \rightarrow L$   $\triangleright$  loss from eq. (14)
15:     $AD(L, \mathbf{w}) \rightarrow \nabla L$   $\triangleright$  sensitivity analysis
16:     $\mathbf{w}, \nabla L \rightarrow \mathbf{w}$   $\triangleright$  Adam optimizer step
17:     $\gamma + \Delta\gamma \rightarrow \gamma$   $\triangleright$  increment penalty
18:    epoch ++
19:  until  $\|\Delta L\| < \Delta L_c^*$  or epoch <
   max_epoch  $\triangleright$  check for convergence
20: end procedure

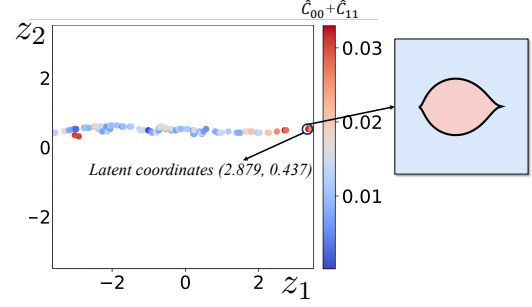
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configuration, and update schemes), we employed the same values across all experiments.

### 3.1 Ideal Microstructure Selection

In this experiment, our objective is to identify a microstructure with a solid volume fraction of approximately 0.25, with the highest permeability. Towards this end, the latent space is uniformly sampled at  $200 \times 200$  points using the decoder. Microstructures with a solid volume fraction within the range  $0.25 \pm 0.001$  are then identified; see fig. 10. Among these, the microstructure with the highest value of the trace of permeability tensor, i.e., highest  $\hat{C}_{00} + \hat{C}_{11}$ , is selected [99]. The chosen microstructure has the following shape parameters  $\mathbf{M}^* = \{a = 0.7158, b = 0.3757, m = 0.6039, n_1 = 1.4787, n_2 = 0.4349, n_3 = 0.5857\}$ . As one can observe in fig. 10, it exhibits a fish-like shape. This particular microstructure will be used in the next numerical experiment.



**Fig. 10:** Latent space coordinates of microstructures with solid volume fraction of approximately 0.25.

### 3.2 Bent-Pipe

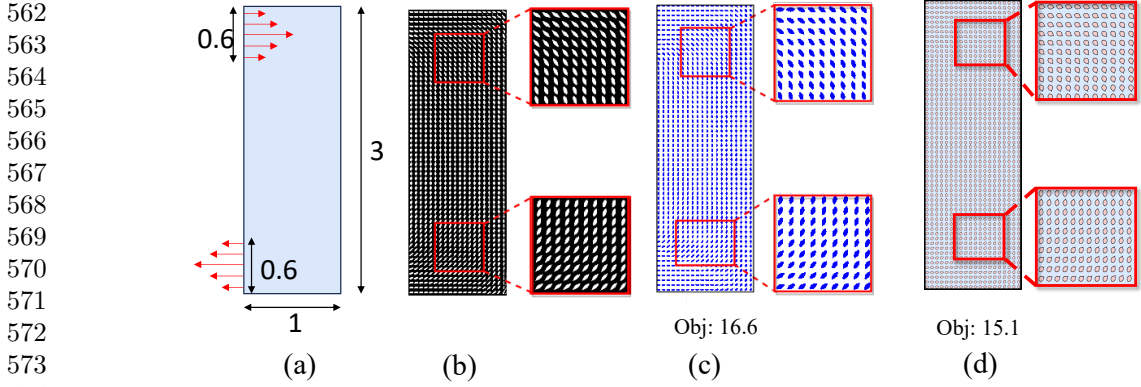
For the remainder of the paper, we set the inlet velocity to 1, the mass density to 1, and viscosity to 1 [25, 100].

We now consider the bent-pipe problem proposed in [31], and illustrated in fig. 11(a). The inlet and outlet boundaries are subject to parabolic velocity conditions, of unit magnitude, and the domain is discretized into  $20 \times 60$  elements. In [31], a two-scale topology optimization was carried out to minimize the dissipated power, with a constraint that the optimal microstructure must occupy exactly 25 percent of each unit cell. The reported topology is illustrated in fig. 11(b); the final dissipated power was not reported. However, as noted in [31], the computed microstructures resemble the fish-body. In [64] a GMTO approach was employed with pre-defined microstructures to achieve a similar design as depicted in fig. 11(c); the dissipated power was reported to be 16.6. Here we use the microstructure selected in the previous experiment to occupy each unit cell. Only the orientation of the microstructure in each cell is optimized. The resulting design is illustrated in fig. 11(d) with the final dissipated power of 15.1. This experiment highlights that super-shapes sampled via the decoder can generate high-performing microstructural designs.

### 3.3 Microstructure Variation

We continue with the previous experiment, but we will now allow the shape and size of microstructures to vary across the domain. A global volume constraint of 0.75 is imposed instead of a unit-cell volume constraint. The resulting design is

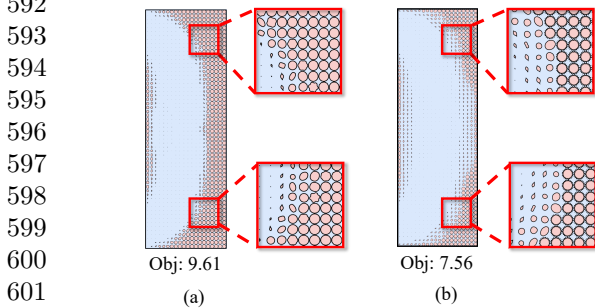
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575 **Fig. 11:** Validation: (a) Problem definition. (b) Solution reported in [31]. (c) Solution reported in [64].  
576 (d) Topology generated via the proposed method.

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579 illustrated in fig. 12a with a dissipated power of  
580 9.61, i.e., the performance improves with increased  
581 design space, as expected. The contact area for  
582 this particular design happens to be 75.69.

583 Finally, instead of imposing a volume constraint,  
584 we impose a contact area constraint of 75.69, and  
585 optimize the design. The final design  
586 is illustrated in fig. 12b, with a dissipated power  
587 of 7.56, i.e., further improvement in performance  
588 is achieved for the same contact area. **The presence  
589 of small solid islands can be attributed to the  
590 absence of penalization in our algorithm.**



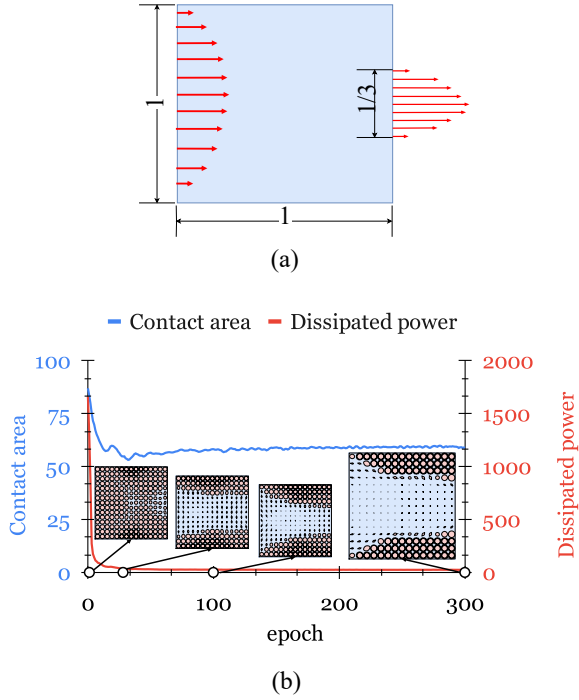
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602 **Fig. 12:** Optimized design with microstructure  
603 variation with: (a) volume constraint, and (b)  
604 contact area constraint.

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608 Note that the dissipated power of 7.56 and  
609 contact area of 75.69 for the design in fig. 12b  
610 are computed using the decoder. For validation,  
611 we re-computed the true values using a global  
612

FEA/homogenization of the final design. The  
dissipated power was found to be 7.87 and the  
contact area was 78.49, i.e., the decoder-  
reconstruction errors are relatively small.

### 3.4 Convergence

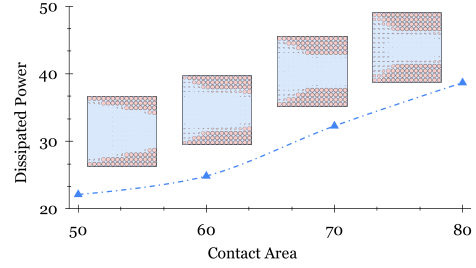
In this experiment, we demonstrate the typical  
convergence of the proposed algorithm using a  
diffuser problem, as shown in Figure fig. 13(a).  
The maximum inlet velocity imposed is 1 unit,  
and the outlet velocity is 3 units. The desired  
contact area was set to 60. The convergence of  
the dissipated power, contact area, and the  
evolving topologies are illustrated in fig. 13(b).  
We observed a stable convergence using the  
simple penalty formulation and Adam optimizer.  
Similar convergence behavior was observed for  
other examples as well.



**Fig. 13:** Convergence of dissipated power and topologies for a diffuser problem. Topologies are illustrated at the 0th, 20th, 100th, and 300th (final) iterations.

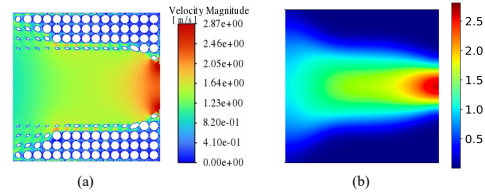
### 3.5 Pareto trade-off

Understanding the trade-off between the objective (dissipated power) and constraint (contact area) through exploration of the Pareto-front is crucial in making informed design choices. In this study, we considered the diffuser problem in fig. 13(a), using the entire design space of microstructures. We computed the optimal topologies for different contact area constraints. Figure 14 illustrates that dissipated power increases with increasing contact area, as expected.



**Fig. 14:** Pareto curve of dissipated power against contact area.

Once again, to determine the accuracy of decoder-reconstruction, we considered the design at the left-bottom corner in fig. 14. For this design, and using the values predicted by the decoder we obtain a dissipated power of 22.13 and a contact area of 50. Further, we reconstructed the microstructures using the shape parameters predicted by the decoder. This is then used to compute the *actual* homogenized matrices. Performing an analysis using these values, we obtain a dissipated power of 22.72 and a contact area of 51.08. This corresponds to an error of 2.6% and 2.9% for the dissipated power and contact area respectively. In addition, for a contact area constraint 60, we compare the velocity magnitude obtained through a full-scale fluid flow simulation using Ansys [101] (see fig. 15(a)), versus the proposed framework (see fig. 15(b)). The maximum velocity magnitude obtained using Ansys is 2.87, whereas using the proposed method results in a value of 2.81.



**Fig. 15:** Velocity magnitude plot of diffuser problem defined in fig. 13(a): (a) using Ansys and (b) proposed homogenization approach.

### 3.6 Computational cost

One of the central hypotheses of this paper is that the proposed offline decoder-based framework offers a significant computational advantage

664 over concurrent homogenization-based optimization.  
 665 Here, we report the computational costs to  
 666 validate this claim.

667 The offline homogenization and data-  
 668 generation of 7000 microstructures (algorithm 1)  
 669 required 164 minutes, while the VAE training  
 670 (algorithm 2) required 90 minutes, i.e., the total  
 671 one-time overhead is around 250 minutes. The  
 672 optimization of the bent-pipe (using a grid size of  
 673  $20 \times 60$ ) took 32 minutes (300 iterations), while  
 674 the optimization of the diffuser (using a grid size  
 675 of  $15 \times 15$ ) took 1.5 minutes (300 iterations).

676 We now consider a hypothetical scenario of  
 677 concurrent homogenization. From the above data,  
 678 observe that the time required for each homog-  
 679 enization is  $164/7000$ , i.e., 1.4 seconds. Now for  
 680 the concurrent homogenization of the bent-pipe,  
 681 one must carry out homogenization over each  
 682 cell within the grid of  $20 \times 60$  over 300 itera-  
 683 tions, the expected optimization time is at least  
 684  $1.4 \times 1200 \times 300/60$ , i.e., 8400 minutes. Similarly,  
 685 for the diffuser, the expected optimization time  
 686 is at least  $1.4 \times 225 \times 300/60$ , i.e., 1575 minutes.  
 687 Thus, the proposed offline decoder-based method  
 688 is computationally far superior.

689

### 690 3.7 Fabrication

691 To demonstrate the manufacturability of the  
 692 designs produced by our framework, we consider  
 693 a design domain with boundary conditions, as  
 694 depicted in fig. 16 (a). In this example, we maintain  
 695 a 1 : 1 ratio and a 3 : 1 ratio between the magni-  
 696 tude of the outlet velocity profiles and the inlet  
 697 velocity profile. Additionally, we enforce a contact  
 698 area constraint of 70, resulting in the design show-  
 699 cased in fig. 16 (b). To ensure that these designs  
 700 can be manufactured successfully, we impose a  
 701 minimum area constraint on each microstructure.  
 702 One can also post-process the obtained design to  
 703 remove microstructures with small volume frac-  
 704 tions (as can be seen in fig. 13(b)). The final 3D  
 705 printed part is illustrated in fig. 16 (c).

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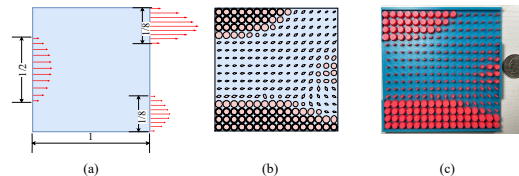


Fig. 16: (a) Design domain with boundary conditions, (b) optimized design, (c) 3d printed design

## 4 Conclusion

In this paper, we presented a novel multi-scale fluid flow topology optimization framework using super-shape microstructures. An offline homogenization, along with the training of a VAE was used to generate a continuous and differentiable latent space of microstructural properties. This was followed by global optimization, where the dissipated power was minimized subject to contact area (or volume) constraint.

The numerical results demonstrate that the proposed method is computationally far superior to concurrent homogenization, with minimal loss in accuracy. Furthermore, super-shapes increase the design space, yielding superior design compared to pre-defined microstructures.

Future research includes extending the framework to (1) high Reynolds flow, (2) thermo-fluid applications, where the contact area is determined indirectly via heat transfer, and (3) structural applications where microstructures with a genus greater than zero are desirable, and connectivity is also critical. Experimental validation, extension to 3D, and imposition of additional manufacturing constraints are also desirable.

## Acknowledgments and Funding Information

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## Statements and Declarations

**Compliance with ethical standards** The authors declare that they have no conflict of interest.

**Replication of Results** The Python code is available at [github.com/UW-ERSL/TOMAS](https://github.com/UW-ERSL/TOMAS). An implementation can also be found in the supplementary section of the paper.

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