

A density-and-strain based K-clustering approach to microstructural topology optimization

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Abstract Microstructural topology optimization (MTO) is the simultaneous optimization of macro-scale topology, and micro-scale structure. MTO holds the promise of enhancing product-performance beyond what is possible today. Furthermore, with the advent of additive manufacturing, the resulting multi-scale structures can be fabricated with relative ease. There are however two significant challenges associated with MTO: (1) high computational cost, and (2) potential loss of microstructural connectivity.

In this paper, a novel density-and-strain based K-means clustering method is proposed to reduce the computational cost of MTO. Further, a rotational degree of freedom is introduced to fully utilize the anisotropic nature of microstructures. Finally, the connectivity issue is addressed through auxiliary finite-element fields. The proposed concepts are illustrated through several numerical examples applied to two-dimensional single-load problems.

Keywords Topology optimization · microstructural optimization design · clustering · principal strain

1 Introduction

Topology optimization is a means of distributing material within a design domain, to optimize performance (Bendsøe and Sigmund (2004); Sigmund and Maute (2013)). It is now a mature field with multitude of methods, including homogenization (Bendsøe and Kikuchi (1988); Hassani and Hinton

(1998)), Solid Isotropic Material with Penalization (SIMP) (Bendsøe (1989)), level set approach (Sethian and Wiegmann (2000); Wang et al. (2003)), topological sensitivity framework (Novotny et al. (2003); Deng and Suresh (2015, 2017)) and evolutionary scheme (Xie and Steven (1993); Yang et al. (1999)). As an example of topology optimization, Fig. 1(a) illustrates a compliance-optimized topology for a structural problem, computed via SIMP.

On the other hand, *microstructural design* is a technique for the distribution of material, at a smaller scale, to optimize material properties. Through microstructural design, one can customize various material behavior (Osanov and Guest (2016)) including bulk/shear modulus (Huang et al. (2011)), Poisson's ratio (Vogiatzis et al. (2017); Xie et al. (2014)), thermal expansion (Sigmund and Torquato (1997)), elasticity tensor (Sigmund (1994)) and other extremal properties (Sigmund (2000)). For example, Fig. 1(b) illustrates an optimal microstructure, once again computed via SIMP, for maximizing shear modulus.

Microstructural topology optimization (MTO) combines topology optimization and microstructural optimization, for simultaneously optimization of topology, at a macro-scale, and microstructures at a smaller-scale. For example, Fig. 1(c) illustrates an optimized MTO design. With the advancement in additive manufacturing (Gao et al. (2015); Liu et al. (2018)), such MTO designs can now be fabricated with relative ease. However, there are several challenges underlying MTO; the objective of this paper is to identify and address some of these challenges.

To understand these challenges, consider a typical MTO problem depicted in Fig. 2 where the objective is to construct an optimal topology, and compute optimal microstructures over each macro finite element. This entails the following: Given an initial topology and an initial set of (random) microstructures, (1) a microstructural analysis (numerical homogenization) is performed for each distinct microstructure,

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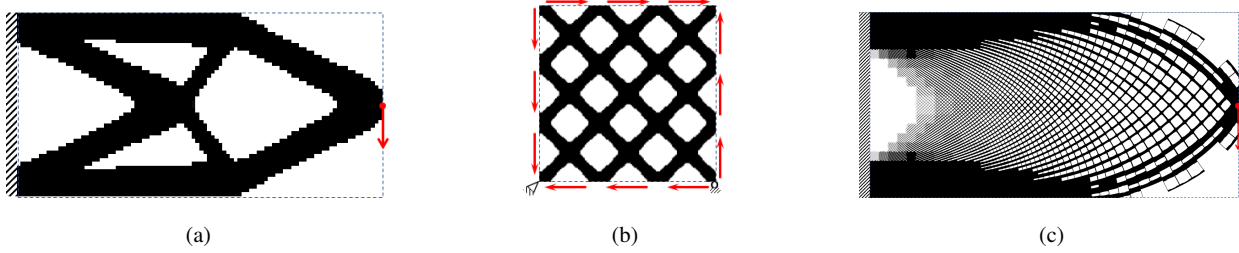


Fig. 1: Different design strategies. (a) Single scale design of short cantilever fixed on the left edge and loaded at the center of right edge, (b) 3×3 units of microstructures optimized for maximum shear stiffness, (c) MTO design using proposed method for same problem description as (a).

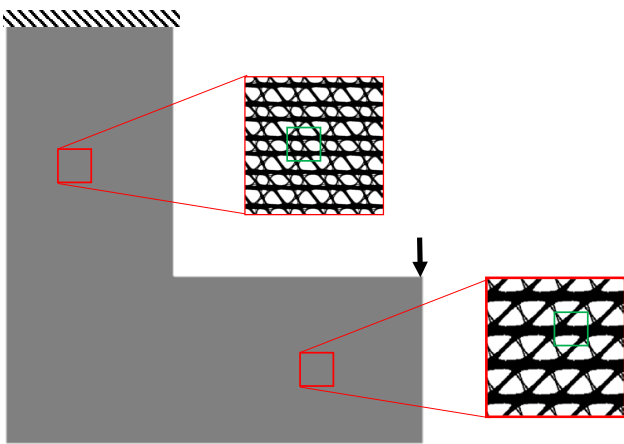


Fig. 2: Schematic of microstructural topology optimization.

to extract the equivalent elasticity tensor (Rodrigues et al. (2002)), (2) the elasticity tensors are then used to assemble a global stiffness matrix, (3) a macroscale analysis is carried out, (4) followed by sensitivity analysis, and (5) finally, the micro and macro design variables are updated, subject to various constraints. These five steps must be repeated numerous times, making MTO “*very demanding*” (Coelho et al. (2008)) and computationally “*quite massive*” (Rodrigues et al. (2002)). Efforts to reduce computational cost can often lead to sub-optimal designs. Further, ensuring topological connectivity between adjacent microstructures is also non-trivial. For example, in Fig. 2, the two microstructures with unit cells are not geometrically compatible when adjacent.

Various strategies have been proposed to address these challenges; these are reviewed in Section 2. In Section 3, we provide a generic MTO formulation. We discuss methods to alleviate the challenges in MTO by introducing *cluster-based MTO design*, which leads to a discussion on sensitivity analysis, and the proposed algorithm in Section 4. In Section 5, several numerical examples illustrate the proposed framework in context of two-dimensional single-load problems. We conclude the paper in Section 6, summarizing the

current work, and suggesting future work with open challenges.

2 Literature review

As mentioned earlier, MTO poses two distinct challenges: (1) high computational cost, and (2) lack of connectivity between microstructures. Various strategies that have been proposed to resolve these challenges are discussed below.

2.1 Strategies to reduce computational cost

A simple strategy, specifically to reduce computational cost, is to constrain all microstructures to be identical. A single microstructure is then controlled by a set of micro design variables, with no macro design variables (Huang et al. (2013)). Due to its simplicity, the computational cost is significantly reduced, and connectivity can be guaranteed. However, the performance of the resulting structure is very poor, i.e., it is often much worse than a classic topology optimized design (Li et al. (2018)).

An extension of the above strategy is to use a single microstructure, but also to include a density variable ρ_e over each macro-element (Liu et al. (2008); Deng et al. (2013); Yan et al. (2014)). Then, the classic SIMP penalization function $f(\rho_e) = \rho_e^p$ can be used to control the presence or absence of each microstructure. The performance improves significantly with little or no additional computational cost. However, an artificial volume constraint must be imposed on the microstructures to avoid trivial solid/void designs. This leads to diminishing performance with decreasing microstructure’s volume fraction (Sivapuram et al. (2016); Deng and Chen (2017)). In other words, under this formulation, the very use of microstructures lowers performance!

A natural generalization is to allow a finite number of microstructures, rather than just one. For example, one can assume that macro-elements with physical proximity share the same microstructural design, leading to the *grid-based clustering* (Sivapuram et al. (2016); Nakshatrala et al. (2013);

Ferrer et al. (2017)). Although the connectivity amongst neighboring microstructures in grid-based clustering can be handled efficiently (Du et al. (2018)), the clustering is far from optimal (as will be demonstrated later on).

Alternately, variable-thickness design optimization (SIMP with $p = 1$) is first performed, and then macro-elements with similar density ρ_e are assumed to have the same microstructural design (Li et al. (2018); Zhang et al. (2018); Liu et al. (2018)), leading to a *density-based clustering*. There are multiple methods for division of macro-elements into clusters. For example, *uniform density clustering* which divides the range $(0, 1]$ into R equal parts (Li et al. (2018)), or *uniform size clustering* where macro-elements sorted according to density are divided into equal sized clusters. Similarly, one can divide macro-elements based on principal stress/strain directions (Xu and Cheng (2018)). K-means clustering (Lloyd (1982)) or k-clustering algorithm identifies already existing clusters in the given distribution of data (Liu et al. (2018)) (e.g. density or principal stress direction) and is likely to perform better than the uniform clustering methods.

In general, with *clustering*, performance improves, with a slight increase in computational cost (depending on the number of clusters).

2.2 Strategies to ensure connectivity

Connectivity has been addressed by several researchers: (1) if the microstructures are graded version of one parent microstructures, connectivity can be easily guaranteed (Wang et al. (2017)), (2) perturbation of coordinates may be used to achieve connected optimized structures (Liu et al. (2017); Zhu et al. (2019)), (3) alternately, passive (non-design) micro-elements can be assigned to all microstructures for connectivity (Zhou and Li (2008); Deng and Chen (2017); Li et al. (2018)), or, (4) a constraint on some connectivity measure can be introduced (Du et al. (2018)), and (5) finally, if the microstructures are simple rotations of rectangular voids, it may be possible to ensure connectivity (Pantz and Trabelsi (2008); Groen and Sigmund (2018); Allaire et al. (2018)). Use of parameterized rectangular voids renders these techniques computationally efficient. Alexandersen and Lazarov (2015) proposed an MTO technique without length-scale separation for better analysis and connectivity at an expense of high computational cost. A similar approach was earlier used by Zhang and Sun (2006) to study length-scale related effect.

2.3 Alternate Strategies

To solve both the computational complexity and the connectivity issue *lattice structures* are used where the topology of the microstructure is fixed, but the lattice parame-

ters are varied to achieve desired properties (Hassani and Hinton (1998); Cramer et al. (2016); Wang et al. (2017)). However, the choice of the lattice topology is often arbitrary, and the performance is typically sub-optimal. Improved performance may be achievable by designing lattice structures on-the-fly (Wang et al. (2017)) or by better choice of lattice for the problem in hand, e.g. rectangular void aligned along principal direction. (Groen and Sigmund (2018); Allaire et al. (2018))

Yet another strategy is the use of *rank-two laminates* (Francfort and Murat (1986); Avellaneda (1987); Jog et al. (1994)) that are layers of solid interspersed with layers of striped solid and void. Since the effective properties of these laminates can be analytically computed, computational cost is reduced significantly. Their optimality for 2-dimensional compliance minimization problem has been demonstrated (Avellaneda (1987); Allaire et al. (2018)). However, these laminates are not manufacturable.

The *free material optimization (FMO)* method has also been proposed (Bendsøe et al. (1994); Kočvara et al. (2008)) where the objective is to first compute optimal elasticity tensors for each macro-element. Once this is complete, the next phase involves computing the microstructures that have the desired elasticity tensors (Schury et al. (2012)). Realizability of the second phase is discussed in (Milton and Cherkaev (1995)). Unfortunately, the second phase can be computationally very demanding, and connectivity is not guaranteed.

2.4 Paper Contributions

In this paper, we show that the performance can be improved significantly and consistently, by clustering elements based on the underlying strain tensor and the SIMP density, and adding a rotational degree of freedom. Connectivity is tackled using an additional finite element analysis. The main contributions of this paper are as follows:

1. Describing various MTO methods, including the proposed one, as special cases of a single generic MTO formulation.
2. While the importance of SIMP density, and strain information for topology optimization, have been identified by Jog et al. (1994); Bendsøe et al. (1994), we justify the importance of these two quantities for MTO clustering, and provide a framework for exploiting them in MTO.
3. Similarly, while the concept of rotational degree of freedom has been identified within the context of microstructural design (Bendsøe and Kikuchi (1988)), we show that this concept can also be exploited for reducing the design space, and for improved clustering.

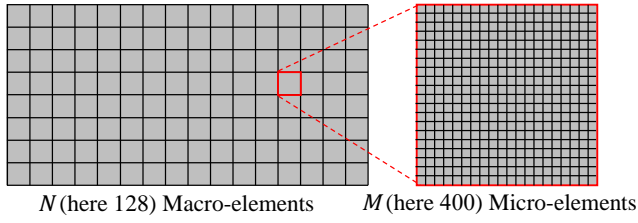


Fig. 3: Illustration of macro and micro elements in a rectangular domain.

3 Microstructural topology optimization (MTO) design

Recall that MTO involves several steps including microstructural analysis, global stiffness matrix assembly, and macro analysis. In this section, the mathematics behind these critical steps is described, together with typical design variables, objective and constraints. Finally, a unified framework that captures popular MTO methods is also presented.

3.1 Design Variables

In MTO, the domain is discretized into N macro-elements, and each macro-element is further divided into M micro-elements as illustrated in Fig. 3. There are typically two sets of (SIMP) design variables (bounded by 0.001 and 1) corresponding to these elements, namely macro and micro variables, resulting in a total of $(N + NM)$ design variables. However, if a clustering strategy is used, several macro-elements are mapped to a single microstructure, and they will share the same set of design variables. Consequently, with clustering, the total number of design variables will be reduced to $(N + RM)$, where R is the number of distinct microstructures. Henceforth, in the presence of clustering, the mapping function from a macro-element n to a microstructure r will be denoted by $r = g(n)$. The macro-variable will be denoted by ρ_n while the micro-variable will be denoted by $\gamma_{r,m}$ corresponding to micro-element m associated with a microstructure r .

3.2 Objective and Constraints

In MTO, the typical objective is to minimize compliance

$$C = u^T K u$$

where K and u are global stiffness matrix and displacement vector, respectively. This is subjected to a global volume constraint:

$$v \sum_{n=1}^N \left(\rho_n \sum_{m=1}^M \gamma_{g(n),m} \right) \leq V^* \quad (1)$$

where v is the volume of a micro-element, and V^* is the desired design volume. An additional volume constraint is often imposed on all microstructures to avoid trivial (fully solid) designs:

$$v \sum_{m=1}^M \gamma_{r,m} \leq v^*, \forall r \in \{1 \dots R\} \quad (2)$$

There are variations to these generic constraints. For example, in the case of a single microstructure, i.e., $R = 1$, the volume constraint on micro-variables is necessarily active. This simplifies Eqn. (1) to $v^* \sum_{n=1}^N \rho_n \leq V^*$, making the global volume constraint independent of micro design variables.

3.3 Microstructure Analysis and Homogenization

An important step in MTO is to compute the homogenized elasticity tensor for each distinct microstructure (Bendsøe and Kikuchi (1988)). First the elasticity tensor of a micro-element m is computed as $D_{r,m} = (\gamma_{r,m})^p D^0$ where D^0 is elasticity tensor of base material, and p is the SIMP penalization factor. The elasticity tensors are then exploited to assemble the microstructure stiffness matrix K_r . Three independent force vectors $f_r^{i=1,2,3}$ corresponding to three unit strains - two normal and one shear strain - are applied as illustrated in Fig. 4. By solving the corresponding three problems $K_r u_r^i = f_r^i$, the homogenized elasticity tensor D_r^H can be extracted (Liu et al. (2002)). Note that these three problems must be solved for each of the R distinct microstructures.

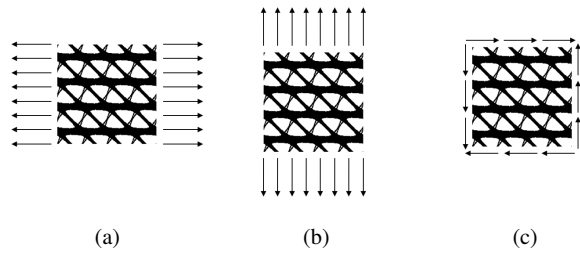


Fig. 4: Homogenization by analyzing microstructure thrice.

3.4 Macro Analysis

Next, the homogenized elasticity tensor D_r^H is either directly used (Rodrigues et al. (2002); Nakshatrala et al. (2013)), or scaled (Liu et al. (2008); Deng et al. (2013); Yan et al. (2014)) via $\rho_n^p D_r^H$, to compute the stiffness matrix of each macro-element. In either case, these macro-element matrices are assembled into a global stiffness matrix K . Finally, the global problem $Ku = f$ is solved, where f is the external

force. The design variables are suitably updated (to be discussed later), and the process is repeated, until convergence is reached.

3.5 Generic Formulation

Given the above definitions, almost all existing MTO (specifically, two scale) formulations can be captured via a generic MTO problem statement as follows.

$$\underset{x}{\text{minimize}} \quad u^T K u \quad (3a)$$

$$\text{subject to} \quad K u = f \quad (3b)$$

$$v \sum_{m=1}^M \gamma_{r,m} \leq v^* \quad \forall r \in \{1 \dots R\} \quad (3c)$$

$$v \sum_{n=1}^N \left(\rho_n \sum_{m=1}^M \gamma_{g(n),m} \right) \leq V^* \quad (3d)$$

$$0.001 \leq x_i \leq 1 \quad \forall i \quad (3e)$$

Various MTO formulations can be interpreted as special cases of the above formulation as in Table 1; references are provided for each formulation type. As one can observe, the design variables x , the constraints, the definition of elasticity matrix for macro-elements, are different for each formulation. Performance with respect to classic topology optimization has also been compared for different methods, where performance less (greater) than 1 implies the MTO formulation performs worse (better) than classic topology optimization (performed using SIMP penalty $p = 3$). Two popular alternatives not included in the table are: (1) variable-thickness design problem which has optimal performance, but violates Hashin-Shtrikman (Hashin and Shtrikman (1962)) bounds and gives non-manufacturable design (Sigmund et al. (2016)), and (2) the use of all design variables (NM) at one scale; this will provide optimized manufacturable design at a prohibitive computational cost. The above formulation can be easily modified for multiple load cases by adding equation for each load case in 3b and adding their contribution to 3a as weighted sum.

Later in the paper we will compare the proposed formulation (discussed next) against some of the formulations in Table 1.

4 Proposed Method

In this section, we discuss the proposed method whose main highlights are: (1) a simplified MTO formulation that eliminates macro design variables and micro volume constraints, (2) a "density-and-strain" based clustering, (3) exploiting rotation variables to increase the design space, while enforcing connectivity.

4.1 Proposed Clustering Method

The proposed MTO formulation is a simplified version of the generic statement in Eqn. (3):

$$\underset{\gamma_1, \gamma_2, \dots, \gamma_R}{\text{minimize}} \quad u^T K u \quad (4a)$$

$$\text{subject to} \quad K u = f \quad (4b)$$

$$v \sum_{n=1}^N \sum_{m=1}^M \gamma_{g(n),m} \leq V^* \quad (4c)$$

$$0.001 \leq \gamma_{r,m} \leq 1 \quad 1 \leq r \leq R, 1 \leq m \leq M \quad (4d)$$

Observe that, in the proposed formulation, there are no macro SIMP variables ρ_n . As a direct consequence, a single global volume constraint (see Eqn. (4c)) is sufficient; eliminating the microstructure volume constraint, which leads to better distribution of material. If $\kappa(r)$ denotes the cardinality of a microstructure r i.e. the number of macro-elements associated with a microstructure r then the volume constraint can be expressed as:

$$v \sum_{r=1}^R \left(\kappa(r) \sum_{m=1}^M \gamma_{r,m} \right) \leq V^* \quad (5)$$

rendering the mapping function $g(n)$ dispensable for imposition of volume constraint.

4.2 Combined density-and-strain based clustering

In Section 2, we reviewed grid-based clustering and density-based clustering. Here, we propose and justify a hypothesis that a "combined density-and-strain" based clustering will lead to better performance. As a motivation, consider the pure density based clustering illustrated in Fig. 5 where a variable thickness design optimization is first carried out; then the resulting density distribution is used to cluster macro-elements. This method however disregards local strain conditions. For example, as illustrated in Fig. 5, the three macro-elements 'a, b and c' with similar densities, but potentially different strain characteristics, are grouped into the same cluster. This, leads to sub-optimal microstructures, as illustrated later on through numerical experiments.

The proposed hypothesis (of strain-and-density based clustering) is further supported by FMO (Bendsøe et al. (1994)) where it is shown that the optimal elasticity components at any location depends both on the local principal strains ϵ_I

Table 1: Various MTO formulations as instances of the generic formulation.

Problem Type	x	Elasticity tensor	(3c)	(3d)	Performance
Classic topology optimization ¹	ρ	$\rho_n^p D^0$	×	✓	1
Single microstructure optimization ²	γ	$D^H(\gamma)$	✓	×	$\ll 1$
Single microstructure + macro-design ³	ρ, γ	$\rho_n^p D^H(\gamma)$	✓	✓	< 1
Unclustered microstructure optimization ⁴	$\gamma_1, \gamma_2, \dots, \gamma_N$	$D^H(\gamma_n)$	✓	×	> 1
Clustered microstructures + macro-design ⁵	$\rho, \gamma_1, \gamma_2, \dots, \gamma_R$	$\rho_n^p D^H(\gamma_r)$	✓	✓	< 1
Clustered microstructures + both constraints ⁶	$\rho, \gamma_1, \gamma_2, \dots, \gamma_R$	$D^H(\gamma_r)$	✓	✓	≈ 1
Clustered microstructures ⁷	$\gamma_1, \gamma_2, \dots, \gamma_R$	$D^H(\gamma_r)$	✓	×	≈ 1

1 - Sigmund (2001), 2 - Huang et al. (2013), 3 - Liu et al. (2008); Deng et al. (2013); Yan et al. (2014), 4 - Rodrigues et al. (2002); Nakshatrala et al. (2013), 5 - Sivapuram et al. (2016), 6 - Li et al. (2018); Zhang et al. (2018), 7 - Nakshatrala et al. (2013); Ferrer et al. (2017); Liu et al. (2018)

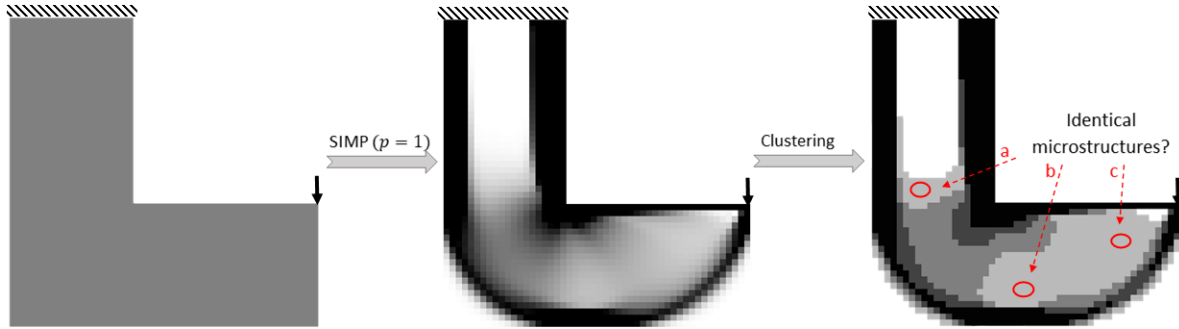


Fig. 5: Schematic of density-based clustering.

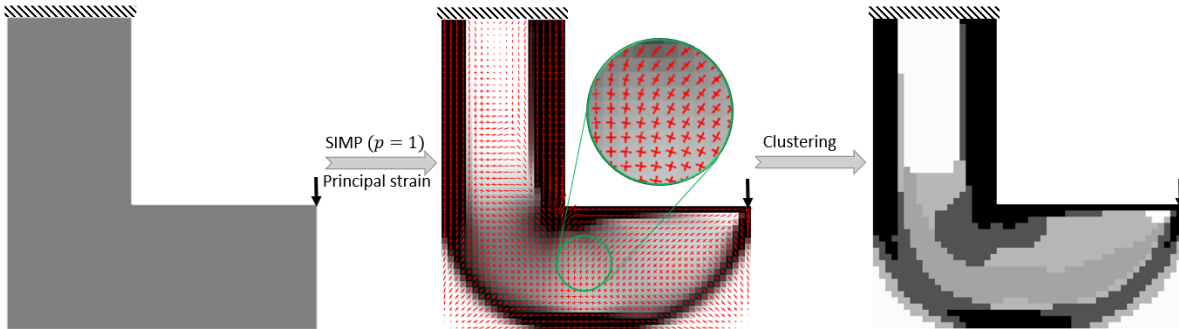


Fig. 6: Schematic of combined density-and-strain based clustering.

and ε_{II} , and the SIMP density ρ , i.e.,

$$D_{11} = \rho \frac{\varepsilon_I^2}{\varepsilon_I^2 + \varepsilon_{II}^2} \quad (6a)$$

$$D_{22} = \rho \frac{\varepsilon_{II}^2}{\varepsilon_I^2 + \varepsilon_{II}^2} \quad (6b)$$

$$D_{12} = \rho \frac{\varepsilon_I \varepsilon_{II}}{\varepsilon_I^2 + \varepsilon_{II}^2} \quad (6c)$$

Further, observe that it is sufficient to consider the strain ratio $\varepsilon_r^2 = \varepsilon_{II}^2 / \varepsilon_I^2$, where $\varepsilon_{II}^2 \leq \varepsilon_I^2$ to ensure $\varepsilon_r \in [0, 1]$. Thus, in this paper, we consider the two scalars $[\rho, \varepsilon_r^2]^T$ as a basis for clustering. In particular, we use Lloyd's K-means clustering algorithm (Lloyd (1982)), together with K-means++ initialization (Arthur and Vassilvitskii (2007)). Note that the

macro-elements with density (obtained via variable thickness design optimization) close to 0, or close to 1, are constrained to two distinct clusters. The combined density-and-strain based clustering is shown in Fig. 6. This clustering can also be performed on designs obtained via SIMP with penalization higher than 1 or via rank-2 laminate (Francfort and Murat (1986)) design so as to not violate Hashin-Shtrikman bounds (Hashin and Shtrikman (1962))

We note that when the principal strains are of opposite signs, optimal periodic microstructure is not achievable (Allaire and Aubry (1999)); also see Fig. 7 of Bendsoe and Sigmund (1999). The proposed clustering scheme leads to sub-optimal designs.

4.3 Microstructure rotation

Observe that, by definition within a given cluster, macro-elements have similar values for the density and principal-strain ratio. However, the principal strain direction can vary significantly within a cluster. Therefore, in order to map the macro-elements within a cluster to a single parent microstructure, they must be rotated by an angle θ_n as determined by the principal direction (see Fig. 7).

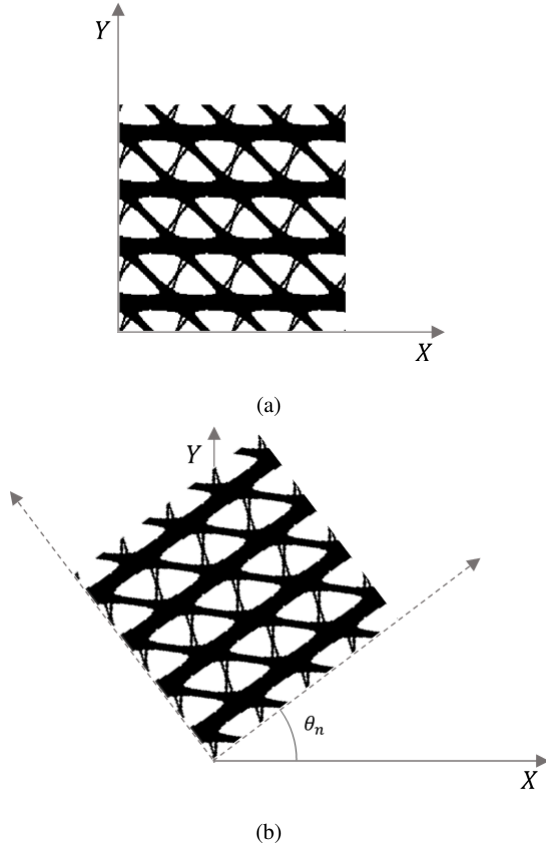


Fig. 7: Rotation of microstructure by angle θ_n .

Rotating microstructures along principal direction (Pedersen (1989)) has two-fold advantage in clustering. The number of strain components is reduced from 3 to 2 (reducing the dimension of the problem), and better performance can be achieved with fewer clusters.

Further, due to this rotation, the elasticity tensor for each macro-element must also be transformed as follows: $D_n = Q(\theta_n)D^H(\gamma_{g(n)})Q^T(\theta_n)$ where rotation matrix $Q(\theta_n)$ is given by

$$Q(\theta_n) = \begin{bmatrix} \cos^2(\theta_n) & \sin^2(\theta_n) & \sin(2\theta_n) \\ \sin^2(\theta_n) & \cos^2(\theta_n) & -\sin(2\theta_n) \\ -\sin(2\theta_n)/2 & \sin(2\theta_n)/2 & \cos(2\theta_n) \end{bmatrix} \quad (7)$$

This amounts to carrying out two matrix-matrix multiplications for each macro-element, during each step of the optimization process, followed by computing the corresponding element stiffness matrix. One can however significantly reduce the computational cost through elemental stiffness matrix K^E templates. Specifically, at the start of the optimization process, the following six templates are computed:

$$\hat{K}_i = K^E(\hat{D}_i) \quad i = 1, \dots, 6 \quad (8)$$

where

$$\begin{aligned} \hat{D}_1 &= \begin{bmatrix} 1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, & \hat{D}_2 &= \begin{bmatrix} 0 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}, & \hat{D}_3 &= \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \\ \hat{D}_4 &= \begin{bmatrix} 0 & 1 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, & \hat{D}_5 &= \begin{bmatrix} 0 & 0 & 1 \\ 0 & 0 & 0 \\ 1 & 0 & 0 \end{bmatrix}, & \hat{D}_6 &= \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 1 \\ 0 & 1 & 0 \end{bmatrix} \end{aligned}$$

Then, during optimization, given a homogenized and transformed D matrix for each macro-element, the corresponding elemental stiffness matrix can be efficiently computed as follows:

$$K^E(D) = D_{11}\hat{K}_1 + D_{22}\hat{K}_2 + D_{33}\hat{K}_3 + D_{12}\hat{K}_4 + D_{13}\hat{K}_5 + D_{23}\hat{K}_6 \quad (9)$$

4.4 Connectivity of microstructures

Finally, we discuss the connectivity issue. Recall that all microstructures within a cluster, map to the same parent microstructure, with possible rotation. If there is no rotation, then the connectivity within the cluster can be easily enforced (see Fig. 8(a)). However, in the presence of rotation, connectivity is typically lost; see Fig. 8(b).

This can be addressed by morphing microstructures (Pantz and Trabelsi (2008); Allaire et al. (2018); Groen and Sigmund (2018)) using a continuously varying coordinate system oriented along principal strain directions. Observe that the strain directions are computed at the center of each macro-cell, and are therefore discontinuous, and must be smoothed. Moreover, the rotational symmetry of the direction field is handled using connected component labelling (Groen and Sigmund (2018)).

Let the desired coordinate transformation be represented by $\tilde{x}(x, y)$ and $\tilde{y}(x, y)$ for every point (x, y) in the design domain. The gradients of this transformation are set equal to the principal strain directions n_1 and n_2 , where

$$n_1 = \begin{bmatrix} \cos(\theta) \\ \sin(\theta) \end{bmatrix}, \quad n_2 = \begin{bmatrix} -\sin(\theta) \\ \cos(\theta) \end{bmatrix}$$

i.e.,

$$\nabla \tilde{x} = n_1, \quad \nabla \tilde{y} = n_2 \quad (10)$$

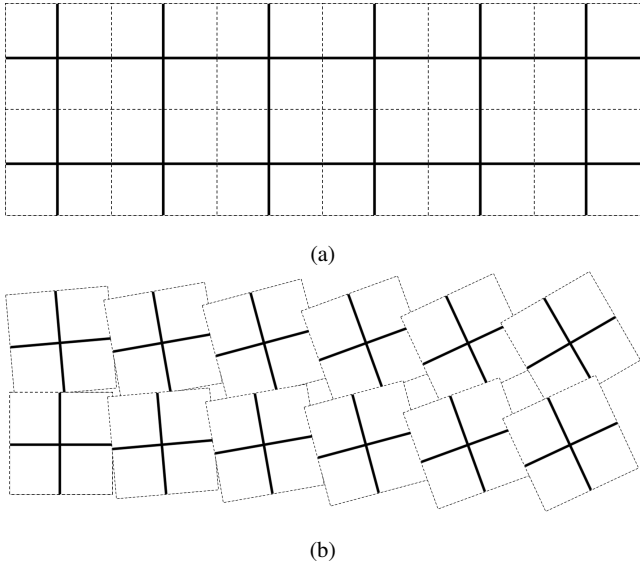


Fig. 8: (a) Arrangement of microstructures without rotation; (b) desired rotations of microstructure.

In theory, these gradients only exist if curl of n_1 and n_2 vanishes everywhere in the domain (Allaire et al. (2018)), but this limitation can be disregarded in practice (see Fig. 13 and its explanation in Allaire et al. (2018)). Prior works by Allaire et al. (2018) and Groen and Sigmund (2018) consider a cosine function formulation that only applies to microstructures with rectangular voids. Here, we consider a variation of this concept that applies to any microstructure.

First, we impose the gradient constraint in a weak sense using a finite element formulation. In other words, the coordinate transformation fields \tilde{x} and \tilde{y} are described using standard finite element basis function N with unknown values \hat{x} and \hat{y} defined at the nodes. The gradient of the basis function is defined as $B = \nabla N$. This leads to a pair of linear systems:

$$\check{K}\hat{x} = \check{F}_1, \quad \check{K}\hat{y} = \check{F}_2 \quad (11)$$

where

$$\check{K} = \int_{\Omega} B^T B d\Omega$$

$$\check{F}_1 = \int_{\Omega} B^T n_1 d\Omega$$

$$\check{F}_2 = \int_{\Omega} B^T n_2 d\Omega$$

The two linear systems have the same stiffness matrix but different force vectors. The use of a finite element formulation used here (and also in Groen et al. (2019) applied to triangle wave function) simplifies the computation, and the resulting fields are free from oscillations as observed in finite difference formulation (Groen and Sigmund (2018)).

Further, unlike the cosine (Allaire et al. (2018); Groen and Sigmund (2018)) or triangle wave (Groen et al. (2019)) formulation, in this paper, the coordinate transformation is used directly to obtain a morphed set of microstructures. Every quad $(\tilde{x}_1, \tilde{y}_1, \tilde{x}_2, \tilde{y}_2)$ such that $(\tilde{x}_2 - \tilde{x}_1 = \Delta x)$, $(\tilde{y}_2 - \tilde{y}_1 = \Delta y)$ is mapped with a microstructure where $(\Delta x, \Delta y)$ is the size of a macro-element. This microstructure $g(n)$ corresponds to the macro-element n with center closest to the center of the quad. These centers are computed in (x, y) as opposed to (\tilde{x}, \tilde{y}) used for quad identification. Fig. 9(a) demonstrates the morphed microstructures generated by using the above concept. The region marked in the figure corresponds to a quad which maps one microstructure.

The above formulation restricts isocontours of \tilde{x} and \tilde{y} to be equispaced, leading to their digression from desired path especially near regions of convergent principal strain directions. This is overcome by having a scalar associated with n_1 and n_2 which dictates the convergence or divergence of isocontours. We choose e^r as suggested by Allaire et al. (2018) and solve for r using an equation identical to Eqn. (10) except that the right hand side becomes $[-\frac{\partial \theta}{\partial y}, \frac{\partial \theta}{\partial x}]^T$ instead of n_1 or n_2 . As we do not eliminate singularities in strain field, which usually occurs in void region, we add weight w which is 0.01 in void regions (Groen and Sigmund (2018)) and one everywhere else. Finite element formulation as described earlier is used first for the solution of $w\nabla r = w[-\frac{\partial \theta}{\partial y}, \frac{\partial \theta}{\partial x}]^T$ followed by the solution of modified Eqn. (10) i.e. $w\nabla \tilde{x} = we^r n_1$ and $w\nabla \tilde{y} = we^r n_2$. Fig. 9(b) illustrates the morphed microstructures generated by using the modified equations.

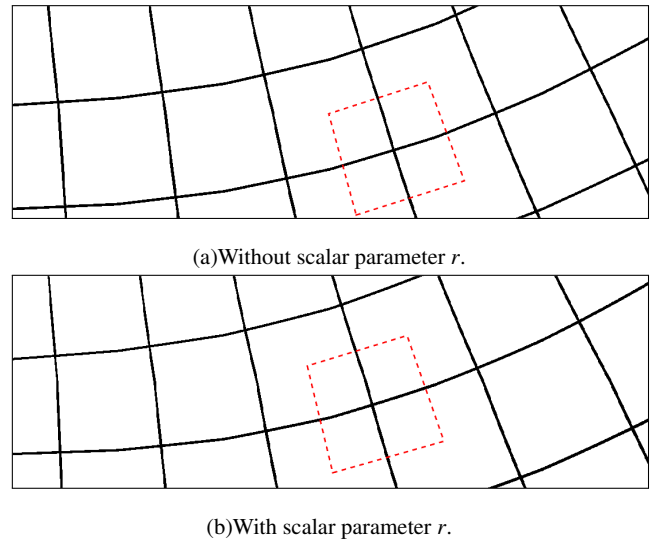


Fig. 9: Morphed microstructures using the continuously varying coordinate system. Highlighted region is mapping of one quad.

As the isocontour spacing is no longer uniform, they must be properly scaled. Based on our experience, a scaling factor of $2\Delta x/|\nabla\bar{x}|$ for \bar{x} and $2\Delta y/|\nabla\bar{y}|$ for \bar{y} is suggested. This factor is problem dependent and needs to be chosen properly to avoid too coarse or too dense quads. Note that even for the same parent microstructure, the size of the quad determines the feature size which varies significantly throughout the domain. Note the variation in the size of microstructures from left (compressed) to right (expanded) in Fig. 9(b) as compared to Fig. 9(a).

After generating the design, a full-scale analysis is performed as a verification step. However, low volume fraction regions with compressed microstructures may exhibit disconnectedness (see Fig.10(a)) which will lead to a singular stiffness matrix. Therefore, a 10x finer discretization of the design domain is used to arrive at final design (as shown in Fig. 10(b)), which is then agglomerated in a mesh with (NM) elements (as shown in Fig. 10(c)) to enable full-scale analysis. In the full-scale analysis, elements penalized with a penalty parameter of 3 to prevent performance over-estimation. (Hashin and Shtrikman (1962)). This two-step approach obviates post-processing of disconnected features.

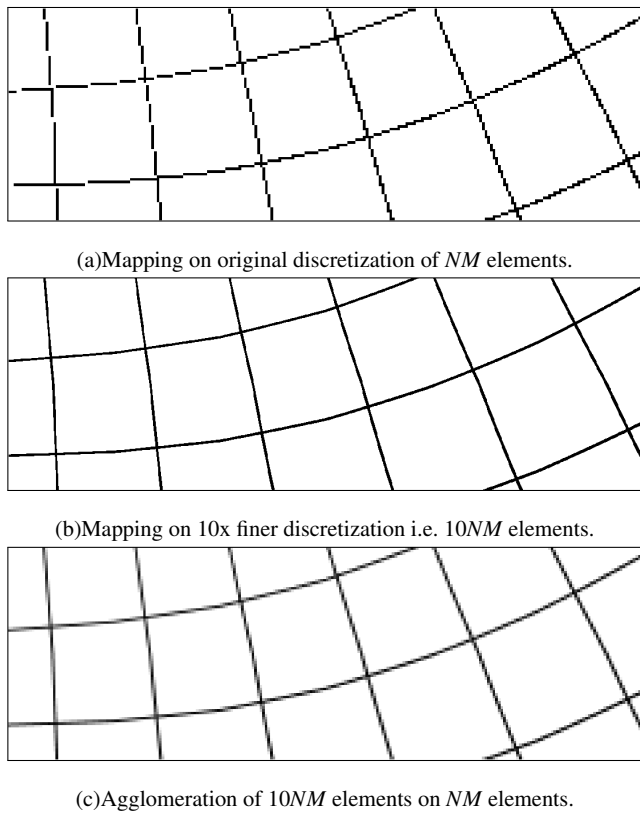


Fig. 10: Agglomeration to ensure connectivity for full-scale analysis

As a final comment on connectivity, observe that microstructures with their axes oriented along principal directions, when optimized, give rise to Vigdergauz-like structures (Vigdergauz (1989, 1994); Bendsøe and Sigmund (1999)) illustrated in Fig. 11. In other words, if a microstructure exists (with non-zero density and principal stress ratio), the boundary of the microstructure is always solid. Thus, we can force the boundary of such microstructures to be a non-design region, as illustrated in Fig. 12. This ensures connectivity between clusters as well. There may be regions not fully solid or void with one principal stress vanishing, then the non-design regions of a microstructure ensure connectivity at the cost of rendering it non-optimal.

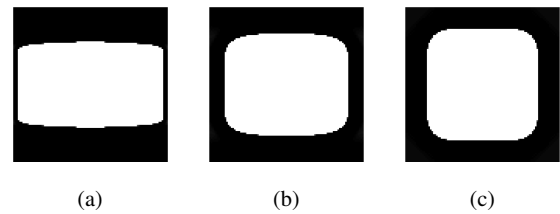


Fig. 11: Microstructure design for volume fraction of 0.5 and principal stress ratio (a) $\frac{\sigma_{II}}{\sigma_I} = 0.1$, (b) $\frac{\sigma_{II}}{\sigma_I} = 0.5$, and (c) $\frac{\sigma_{II}}{\sigma_I} = 1$.

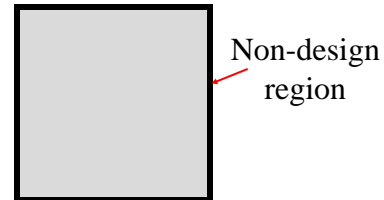


Fig. 12: Proposed non-design elements (shown in dark color) for ensuring connectivity.

4.5 Sensitivity Analysis and Design Update

We now develop the sensitivity equations; the sensitivity of compliance C with respect to the micro-design variable $\gamma_{r,m}$ is given by:

$$\frac{\partial C}{\partial \gamma_{r,m}} = u^\top \frac{\partial K}{\partial \gamma_{r,m}} u + 2u^\top K \frac{\partial u}{\partial \gamma_{r,m}} \quad (12)$$

where the symmetry of global stiffness matrix K has been exploited. We will assume that the load is design independent. Therefore, from the governing equation $Ku = f$, we have

$$K \frac{\partial u}{\partial \gamma_{r,m}} = -\frac{\partial K}{\partial \gamma_{r,m}} u$$

Substituting this in Eqn. (12) leads to

$$\frac{\partial C}{\partial \gamma_{r,m}} = -u^T \frac{\partial K}{\partial \gamma_{r,m}} u \quad (13)$$

One can express this as a sum over all the macro-elements

$$\frac{\partial C}{\partial \gamma_{r,m}} = -\sum_{n=1}^N u_n^T \frac{\partial K_n^E}{\partial \gamma_{r,m}} u_n \quad (14)$$

Exploiting Eqn. (9), the derivative of an elemental stiffness matrix can be expressed as

$$\begin{aligned} \frac{\partial K_n^E}{\partial \gamma_{r,m}} = & \hat{K}_1 \left(\frac{\partial D_n}{\partial \gamma_{r,m}} \right)_{11} + \hat{K}_2 \left(\frac{\partial D_n}{\partial \gamma_{r,m}} \right)_{22} \\ & + \hat{K}_3 \left(\frac{\partial D_n}{\partial \gamma_{r,m}} \right)_{33} + \hat{K}_4 \left(\frac{\partial D_n}{\partial \gamma_{r,m}} \right)_{12} \\ & + \hat{K}_5 \left(\frac{\partial D_n}{\partial \gamma_{r,m}} \right)_{13} + \hat{K}_6 \left(\frac{\partial D_n}{\partial \gamma_{r,m}} \right)_{23} \end{aligned} \quad (15)$$

where $D_n = Q(\theta_n) D_r^H(\gamma_r) Q^T(\theta_n)$. Finally, due to the mapping

$$\frac{\partial D_n}{\partial \gamma_{r,m}} = \begin{cases} Q(\theta_n) \frac{\partial D_r^H}{\partial \gamma_{r,m}} Q^T(\theta_n), & \text{if } r = g(n) \\ 0, & \text{otherwise} \end{cases} \quad (16)$$

Sensitivity of homogenized elasticity tensor is derived using the periodicity of microstructure (Liu et al. (2002)). The sensitivity of the volume constraint in Eqn. (5) is given by $\nu \kappa(r)$.

To avoid checker-boarding (Jog et al. (1994)), the following sensitivity filter (Sigmund (2007)) is used

$$\widetilde{\frac{\partial C}{\partial \gamma_{r,m}}} = \frac{1}{\gamma_{r,m}} \sum_{i \in \Theta_m} W_{mi} \gamma_{r,i} \frac{\partial C}{\partial \gamma_{r,i}} \quad (17)$$

where W_{mi} is cone type signed distance function, and Θ_m is the set of neighboring micro-elements. Note that the filters used for MTO must account for periodicity of microstructures (Xu and Cheng (2018)). Further, in order to obtain solid/void designs, the filter radius is decreased from 4 to 1 (relative to micro-element width), in steps of 0.5, each time the relative change in compliance drops below 1×10^{-4} in two consecutive iteration; this is an accepted practice in SIMP to arrive at solid/void designs (Sigmund (2007)).

The ratio of sensitivities of objective and volume constraint is contained in the term

$$B_{r,m}^{(i)} = - \left(\left(\frac{\widetilde{\frac{\partial C}{\partial \gamma_{r,m}}}}{\gamma_{r,m}} \right)^{(i)} / \Lambda^{(i)} \nu \kappa(r) \right)^\eta \quad (18)$$

where $\Lambda^{(i)}$ is the Lagrange multiplier corresponding to the volume constraint, η is the damping parameter set to 0.5, and i is current iteration index. The design update is carried

out using the well-known optimality criteria method (Sigmund (2001)) utilizing Eqn. (18)

$$\gamma_{r,m}^{(i+1)} = \begin{cases} \max(\gamma_{r,m}^{(i)} - \zeta, \underline{\gamma}) & \text{if } \gamma_{r,m}^{(i)} B_{r,m}^{(i)} \leq \max(\gamma_{r,m}^{(i)} - \zeta, \underline{\gamma}) \\ \min(\gamma_{r,m}^{(i)} + \zeta, 1) & \text{if } \min(\gamma_{r,m}^{(i)} + \zeta, 1) \leq \gamma_{r,m}^{(i)} B_{r,m}^{(i)} \\ \gamma_{r,m}^{(i)} B_{r,m}^{(i)} & \text{otherwise} \end{cases} \quad (19)$$

As is well known, microstructural designs are non-unique; a small move parameter ζ prevents multiple holes appearing and disappearing during optimization, improving stability; here, the move parameter is set to 0.05. The lower limit $\underline{\gamma}$ on design variables is 0.001. The Lagrange multiplier $\Lambda^{(i)}$ is updated in an inner loop using binary search to satisfy the volume constraint Sigmund (2001).

4.6 Algorithm

A flowchart depicting the proposed algorithm is illustrated in Fig. 13. The algorithm can be divided into three distinct phases, and these are described below.

The first phase starts with initialization and discretization of design domain with macro and micro elements. Then, variable-thickness design optimization is performed over the macro-elements. This is followed by computation of principal strain ratio, and principal directions. The SIMP density as well as the principal strain ratio, are exploited to perform K-clustering. This divides the macro-elements into R clusters. The micro density variables for each of the R clusters are initialized as in Fig. 14, where the difference between the two regions is 0.2, while the mean density of the microstructures is matched with the SIMP density. This ensures that the sensitivity values do not vanish everywhere.

In the second phase, homogenization carried out for each of the R microstructures, followed by macro-analysis. The sensitivity is computed and filtered using Eqn. (17); the design variables are updated as described earlier. This process is repeated until either the change in micro-variables is small (here, 0.005), or the relative change in objective is small (here, 5×10^{-6}).

When the optimization terminates, the algorithm enters the third phase where the microstructures are morphed and mapped on a ten times finer mesh to ensure connectivity. Then, a final analysis of the mapped design is performed by agglomeration, as described earlier. In the following examples, the compliance values obtained using full-scale analysis are reported in parenthesis along with the values obtained using homogenization approach.

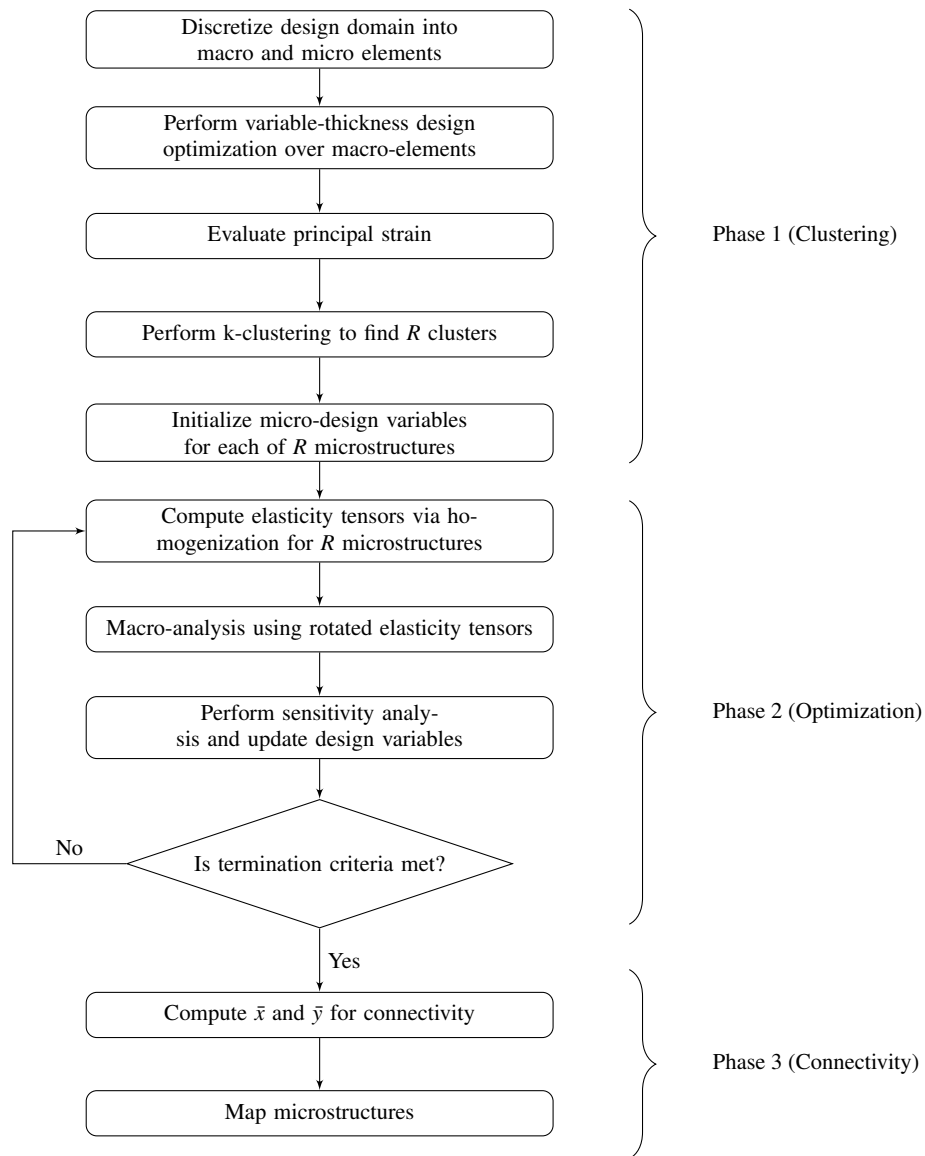


Fig. 13: Flowchart of proposed MTO algorithm.



Fig. 14: Microstructure initialization.

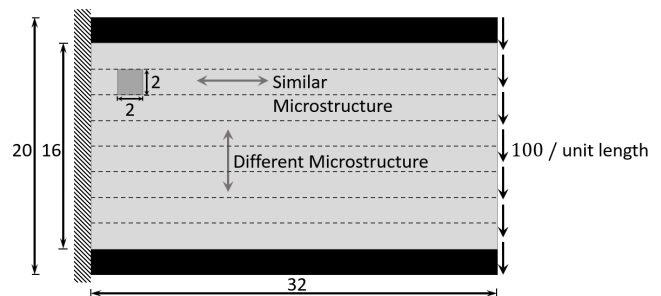


Fig. 15: Design domain and microstructure from Zhang and Sun (2006).

5 Numerical examples

5.1 Microstructural design validation

We consider an example from Zhang and Sun (2006) where the rectangular design domain, illustrated in Fig. 15, is fixed

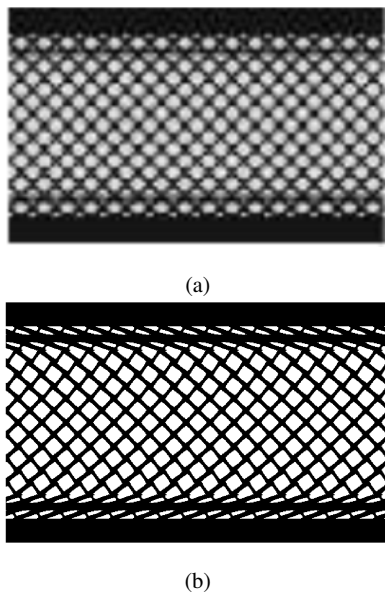


Fig. 16: Design comparison: (a) as illustrated in Zhang and Sun (2006), and (b) obtained via proposed method.

on the left edge and a uniform shear load is applied on the right edge. The domain is discretized into 16×10 (macro) elements which are then clustered into 10 different horizontal layers. The top and bottom layers are made solid as in Zhang and Sun (2006), and the remaining layers have a distinct microstructure, leading to a total of 8 different microstructures (using grid based clustering to be consistent with the published result in Zhang and Sun (2006)). Each macro-element is divided into 1600 micro-elements. The Young's modulus is 1000, and Poisson's ratio is 0.3. No connectivity is enforced among different microstructures. The compliance is minimized for an overall volume fraction of 0.6.

The design obtained by Zhang and Sun (2006) is illustrated in Fig. 16(a), while the design obtained via the proposed method is illustrated in Fig. 16(b). The compliance for the design in Zhang and Sun (2006) was reported as 172171.8, while the compliance of the proposed design is 145583. For comparison, a full-scale analysis was performed on the final design with 0.25 million elements, resulting in a compliance of 147806. Thus, in this case, homogenization leads to an error of less than 2%. The difference can be attributed to the filtering method; the method used here leads to a distinct solid/void design, resulting in a lower compliance value. No finer mesh mapping is required because of the absence of any morphing in this example.

5.2 Comparison of clustering methods

We will now compare the proposed density-and-strain based clustering against grid-based clustering, and density-based clustering. Observe that the clustering method only affects phase 1 of the algorithm. The design domain is an L-bracket (see Fig. 17(a)), discretized into 1600 macro-elements where each macro-element is further divided into 1600 micro-elements. The Young's modulus is 10 and Poisson's ratio is 0.3; the number of clusters R is chosen to be 12 and desired global volume fraction is 0.5.

For grid based clustering, adapting from Sivapuram et al. (2016), the domain is divided into 12 regions as illustrated in Fig. 17(b). The optimized design is given in Fig. 17(c). For density-based clustering, first, variable-thickness design optimization is carried out (see Fig. 17(d)). This is used to form clusters as illustrated in Fig. 17(e), resulting in the design shown in Fig. 17(f). Finally, using the proposed density-and-strain based method, the clusters are formed as illustrated in Fig. 17(h), while Fig. 17(i) illustrates the final design.

The convergence plots for the three cases are illustrated in Fig. 18. The staircase shape obtained in convergence history is due to the change in sensitivity filter radius. The compliances and computational costs are summarized in Table 2. As expected, the compliance from the proposed method is

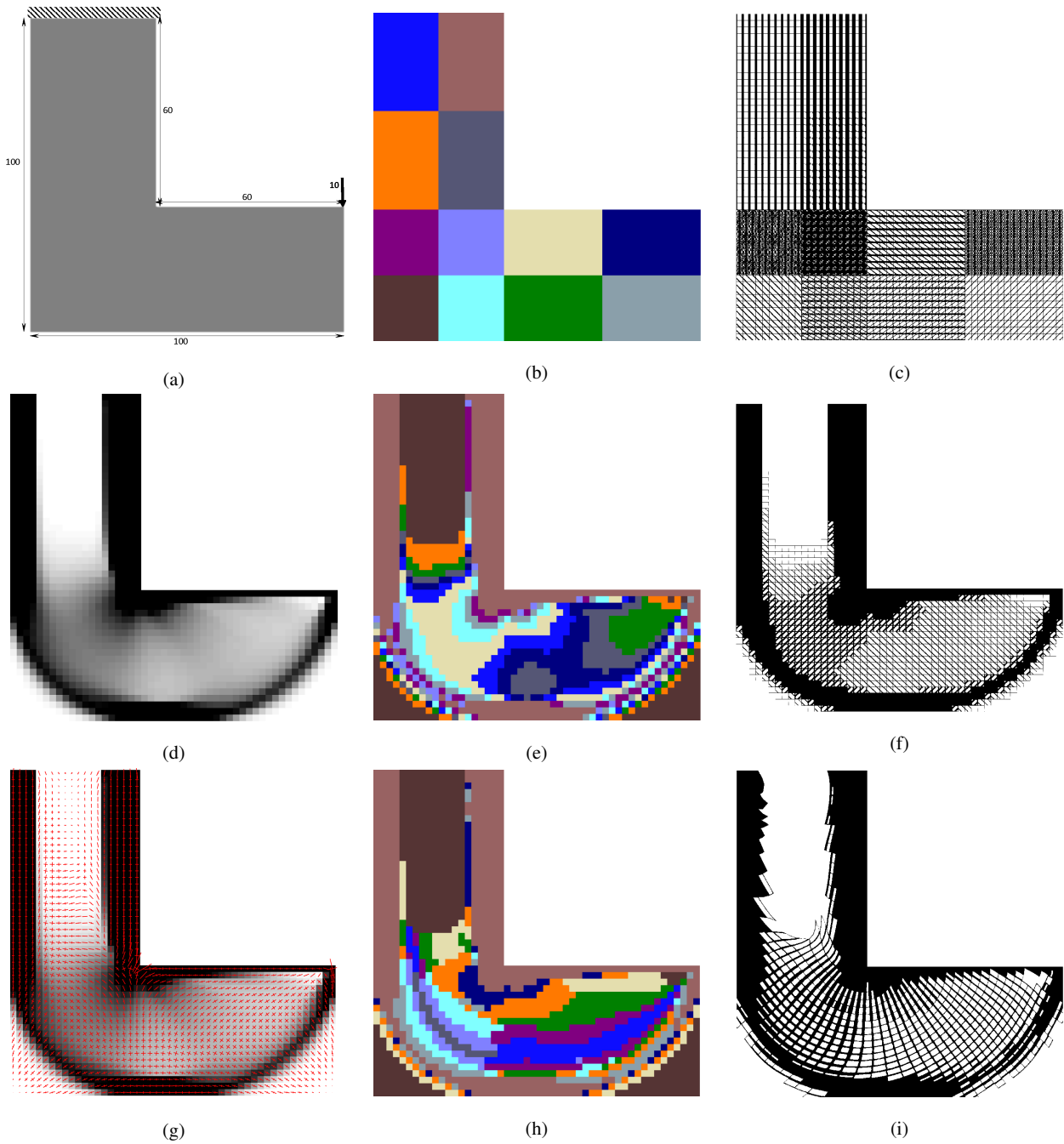


Fig. 17: L-bracket to perform cluster based MTO. (a) Design domain; grid based clustering: (b) clusters and (c) design; density based clustering: (d) density from variable-thickness design optimization, (e) clusters, and (f) design; combined density-and-strain based clustering: (g) density from variable-thickness design optimization and principal strain, (h) clusters, and (i) design.

Table 2: L-bracket:final objective and computation time; the quantity within parenthesis is obtained by a full-scale analysis of the final design, without homogenization.

Clustering Type	Final Objective	Iterations	Computation Time (s)	Avg. Time per Iteration (s)
Grid	3096.4 (3238.7)	115	592	6.1
Density	1936.4 (2086.50)	67	424	6.3
Density-and-strain	1717.3 (2056.02)	48	586	12.2

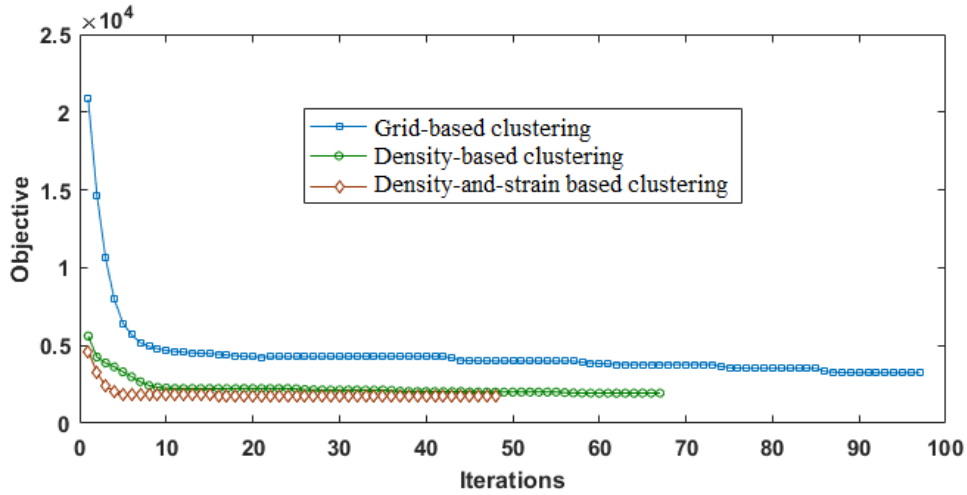


Fig. 18: Convergence history for the three types of clustering.

better than the other two using homogenization with no significant increase in computational cost. The mesh for full scale analysis consists of 2.56 million elements.

For further validation, consider a bridge problem shown in Fig. 19(a). The Young's modulus is 2.1×10^5 and Poisson's ratio 0.3. The design domain is shown in Fig. 19(b) having a point force of 1000, whereas the contours of \bar{x} and \bar{y} along with the principal directions are shown in figs. 19(c) to 19(e). The optimization results are shown for density (see fig. 19(h)) and density-and-strain based (see fig. 19(k)) clustering methods with objective values 418.08 (490.53) and 342.26 (466.95), respectively. The improved compliance values in these two examples validate the superiority of density-and-strain based clustering over density based clustering. The full scale analysis is performed on a 800×1600 mesh.

5.3 Effect of number of clusters and comparison against SIMP design

Here, we compare and show the benefit of proposed method, against variable thickness and SIMP ($p = 3$) designs; we will also study the effect of the number of clusters. As an example, we consider a short cantilever beam with mid-edge load on the right edge illustrated in Fig. 20. The domain is discretized into 800 macro-elements, where each macro-element is subdivided into 1600 micro-elements. The Young's modulus is 2.1×10^5 , Poisson's ratio is 0.3, and desired volume fraction is 0.5.

The designs from variable-thickness and SIMP ($p = 3$) optimization performed on macro-element discretization, are illustrated in Fig. 21(a) and 21(b), respectively. Designs obtained for various choices of number of clusters using the proposed method are illustrated in the Figures 21(c) to 21(f).

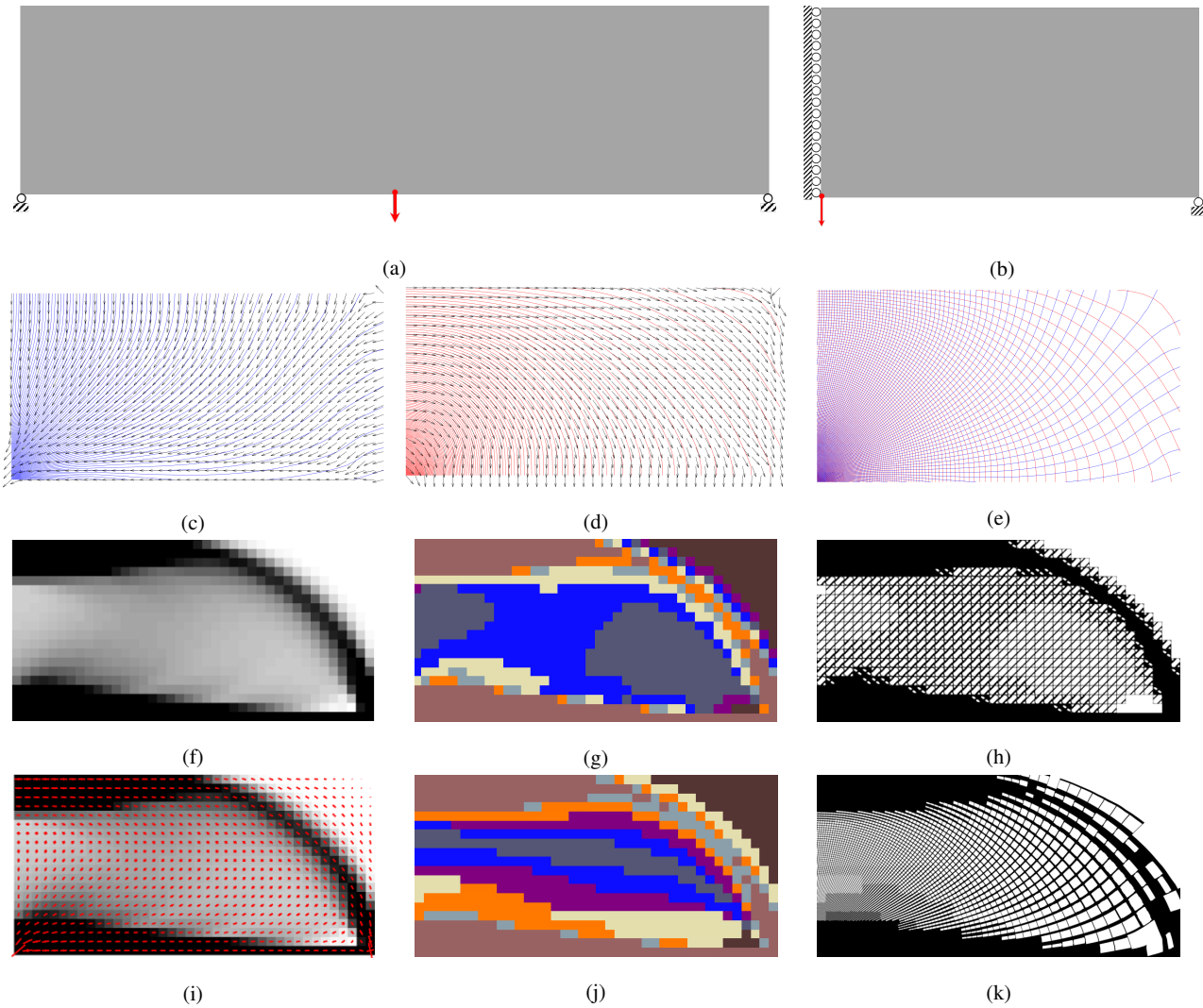


Fig. 19: Bridge problem: (a) problem domain, (b) design domain; Auxiliary fields: (c) direction n_1 and \bar{x} , (d) direction n_2 and \bar{y} , (e) \bar{x} and \bar{y} ; density based clustering: (f) density from variable-thickness design optimization, (g) clusters, and (h) design; combined density-and-strain based clustering: (i) density from variable-thickness design optimization and principal strain, (j) clusters, and (k) design.



Fig. 20: Short cantilever fixed on the left edge and loaded on the center of right edge. Length of line of load = 8.

Table 3 summarizes the results for different cases. An increase in compliance in full-scale analysis (performed on a

800×1600 mesh) is expected since we are distorting the generated microstructures. However, the large disparity in the two numbers may be attributed to a lot of gray elements and imperfect morphing, which requires further work. Notice that the increase in number of clusters enhances the performance, but it does not approach the variable thickness design or surpass SIMP ($p = 3$) design in full-scale analysis. This is attributed to clustering performed only once at the start of the optimization process. Dynamic clustering may lead to improved performance, and will be pursued in future. Further, the poor performance for 3 cluster case is due to the assignment of all the macro-elements to one cluster except for those assigned to the two constrained (with variable thickness design density close to 0 or 1) clusters. As expected, the average time per iteration increases with in-

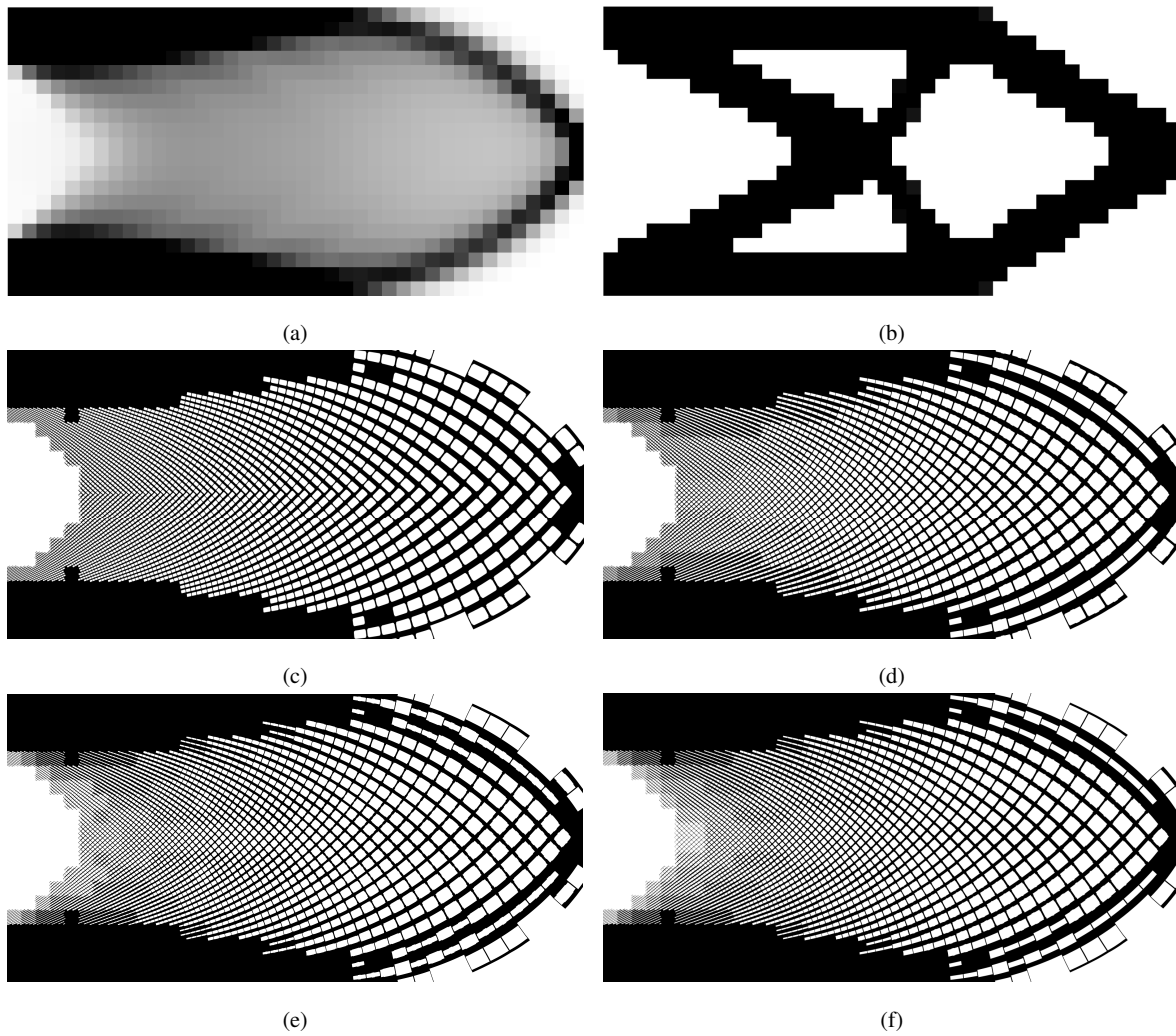


Fig. 21: (a) Variable-thickness design (b) SIMP ($p = 3$) design. Designs with different number of clusters: (c) $R = 3$, (d) $R = 5$, (e) $R = 10$, and (f) $R = 15$.

Table 3: Compliance for short cantilever beam with mid-edge load.

Figure	Type	Number of Clusters	Compliance	Iterations	Avg. time per iteration (s)
21(a)	Variable thickness design	-	10.335	82	0.03
21(b)	SIMP ($p = 3$) design	-	11.86	51	0.06
21(c)	Clustered design	3	12.07 (14.10)	49	5.57
21(d)	Clustered design	5	11.49 (13.0)	49	5.73
21(e)	Clustered design	10	11.38 (12.84)	58	5.82
21(f)	Clustered design	15	11.34 (12.74)	50	5.96

crease in the number of clusters. In case of no-clustering i.e. clusters equal to number of macro-elements (800 in this case), the average time per iteration blows up to 22 minutes.

6 Conclusion

In this paper, a generic MTO formulation is provided which encompasses most of the MTO problems in published literature. The challenges associated with MTO such as large

computational cost and loss of connectivity due to the presence of microstructures are tackled in the proposed method. This comprises of three distinct phases: clustering, optimization and connectivity. In the first phase, a novel combined density-and-strain based clustering method is proposed to divide macro-elements into a pre-defined number of clusters. During the next phase, rotated microstructures are optimized for minimizing the compliance. These rotations are based on principal directions. In the final phase, connectiv-

ity of rotated microstructures are ensured by solving a pair of linear system to define a transformation field. This field is used to project computed microstructures into the design domain.

The proposed clustering method is demonstrated to perform well for small number of clusters. Its superiority as compared to other clustering methods has been demonstrated. K-clustering is used to tackle clustering of two-dimensional parameter involving density ρ and principal strain ratio $(\varepsilon_{II}/\varepsilon_I)^2$. The anisotropic nature of microstructures are effectively utilized by adding the rotational degree of freedom. A simpler method of handling microstructure connectivity for rotated microstructures is proposed as compared to existing methods. The effectiveness of proposed method is also manifested by fewer number of iterations required for convergence.

There are certain aspects of the work which needs to be pursued in future. Starting with a dynamic clustering scheme to redistribute clusters according to evolving design and strain field, which is likely to improve the performance further. A smooth variation is not obtained in the presence of singularity in strain directions as shown by Allaire et al. (2018) and therefore, the auxiliary coordinate system distorts in the neighborhood of singularity. An approach to identify and handle these singularities will be dealt in future along with extending this work to 3D and for objectives other than compliance. Since density-and-strain based clustering and principal-strain based rotation may not be applicable to non-compliance problems, alternate strategies need to be explored.

Replication of results

The MATLAB code and data files required to replicate results presented here, are available at http://www.ersl.wisc.edu/software/MTO_Code.zip.

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